

MECHANICS OF SOLIDS AND MATERIALS 2013-2014

2° Term. Class started on [2014-02-24], room A1.3

(1-2)₁

16:00-18:00

Very small body: body point and its positions

$$p: \{A\} \times \mathbb{R} \rightarrow \mathcal{E}$$

Collection of body points

$$B = \{A, B, C\}$$

placement

$$p: \{A, B, C\} \rightarrow \mathcal{E}$$

motion

$$p: \{A, B, C\} \times \mathbb{R} \rightarrow \mathcal{E}$$

one-parameter family of placements

trajectories, trajectories

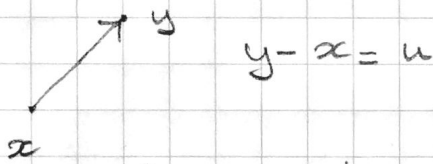
shapes

$$\text{imp} \subset \mathcal{E}$$

Operations on positions

$$x + u = y$$

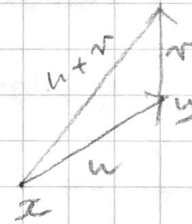
$$\left\{ \mathcal{E}, \mathcal{V} \right\} \begin{array}{l} \text{translation space} \\ \text{Euclidean space} \end{array}$$



for any two positions (x, y) there is a vector u translation x to y ; this vector is unique.

$$(x+u)+v = x+(u+v) \Rightarrow$$

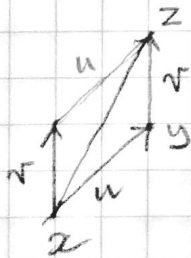
$$x+0 = x$$



$$x+u = y \Rightarrow y-x = u$$

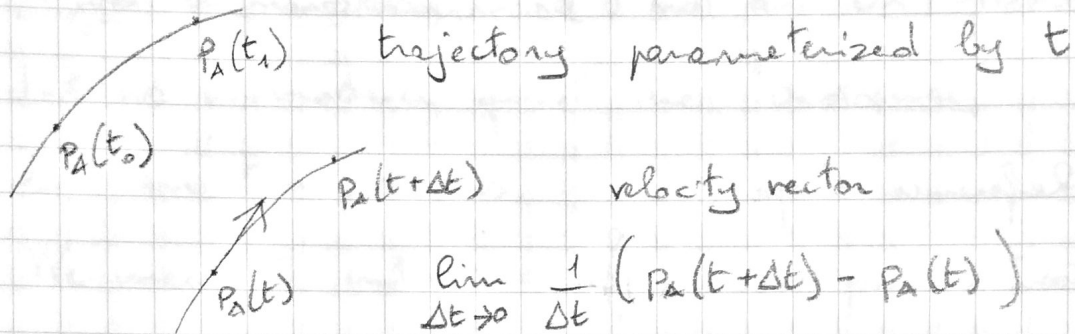
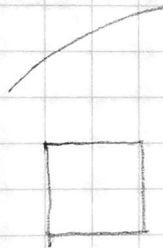
$$(x+u)+(-u) = x+0 = x$$

$$y+(-u) = x \Rightarrow x-y = -u$$



$$\begin{aligned} (x+u)+v &= x+(u+v) \\ &= x+(v+u) = (x+v)+u \end{aligned}$$

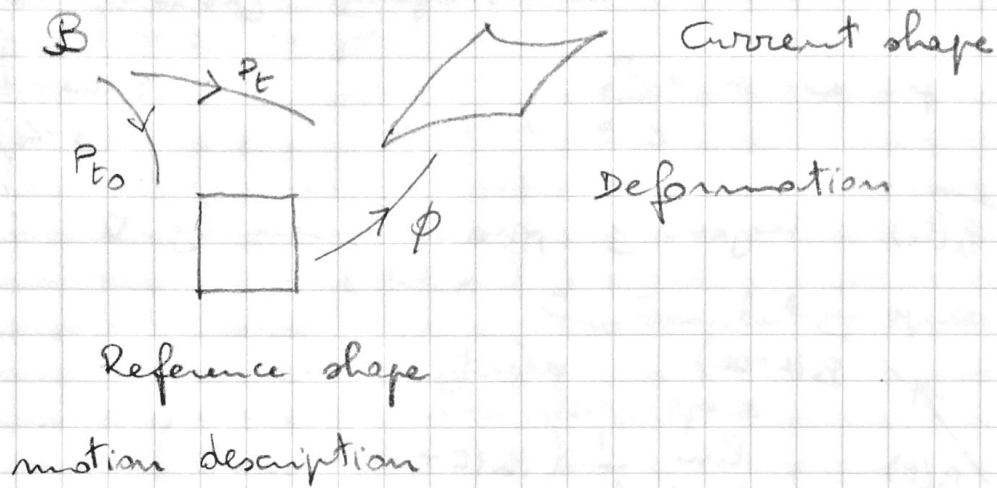
Tuesday [2014-02-25]

motion $p: \mathcal{B} \times \mathbb{R} \rightarrow \mathcal{E}$ 9:00 - 11:00
(3-4)₁Curve-like shape parameterized by a scalar s Square-like shape parameterized by two scalars s_1, s_2 

$$\hat{x}: [0, l] \rightarrow p_c(\mathcal{B})$$

$$\hat{x}: [0, l_1] \times [0, l_2] \rightarrow p_c(\mathcal{B})$$

Dimension of the shape = Dimension of the body



$$\phi : p_{t_0}(\mathcal{B}) \times \mathbb{R} \rightarrow \mathcal{E}$$

$$\bar{\mathcal{R}}$$

$$\phi(\bar{\mathcal{R}}, t) = \mathcal{R}_t$$

"region" of \mathcal{E}

We will discover properties of a deformation
by comparing shapes

Wednesday [2014-02-26]

(5-6), 11:00 - 13:00

Rigid deformation (it's not an OXYMORON)

let us introduce first the "distance" between two portions.

We need to introduce a scalar product \cdot and the induced norm.

The distance between any two portions is left unchanged by a rigid deformation.

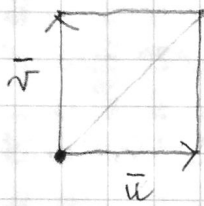
$$\|P_A - P_B\| = \|\bar{P}_A - \bar{P}_B\|$$

$$\begin{aligned} \|u + v\|^2 &= (u+v) \cdot (u+v) = u \cdot u + 2u \cdot v + v \cdot v \\ &= \|u\|^2 + 2u \cdot v + \|v\|^2 \end{aligned}$$

$$\|\bar{u} + \bar{v}\|^2 = \|\bar{u}\|^2 + 2\bar{u} \cdot \bar{v} + \|\bar{v}\|^2$$

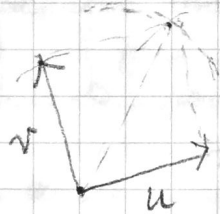
Subtracting we get $0 = u \cdot v - \bar{u} \cdot \bar{v}$

He the scalar product of any two vectors is left unchanged



$$\bar{u} \mapsto u, \quad \bar{v} \mapsto v$$

$$\bar{u} \cdot \bar{v} = 0 \Rightarrow u \cdot v = 0$$



$$\bar{u} + \bar{v} \mapsto u + v \quad \text{linearity}$$

$$u = R\bar{u}, \quad v = R\bar{v}$$

$$u \cdot v = \bar{u} \cdot \bar{v} \Rightarrow R\bar{u} \cdot R\bar{v} = \bar{u} \cdot \bar{v}$$

$$R^T R \bar{u} \cdot \bar{v} = \bar{u} \cdot \bar{v}$$

$$(R^T R - I) \bar{u} \cdot \bar{v} = 0$$

Rigid deformation representation

$$\phi(\bar{p}_A) = \phi(\bar{p}_0) + R(\bar{p}_A - \bar{p}_0)$$

Rigid motion

$$\phi(\bar{p}_A, t) = \phi(\bar{p}_0, t) + R(t)(\bar{p}_A - \bar{p}_0)$$

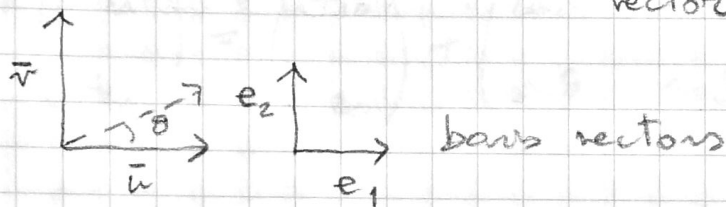
(7-8)₂ Monday [2014-03-03]

16-18 Δ1.3

Matrix representation of the rotation tensor
and coordinate representation of a rigid
deformation

$$\phi(\bar{P}_A) = \phi(\bar{P}_0) + R(\bar{P}_A - \bar{P}_0)$$

↑
vectors



$$R\bar{u} = \cos\theta \bar{u} + \sin\theta \bar{v}$$

$$R\bar{v} = -\sin\theta \bar{u} + \cos\theta \bar{v}$$

$$R\bar{u} \cdot R\bar{u} = \cos^2\theta \bar{u} \cdot \bar{u} + \sin^2\theta \bar{v} \cdot \bar{v} + 2\cos\theta \sin\theta \bar{u} \cdot \bar{v}$$

$$\text{if } \bar{u} \cdot \bar{v} = 0$$

$$\bar{u} \cdot \bar{u} = \bar{v} \cdot \bar{v}$$

$$\Rightarrow$$

$$R\bar{u} \cdot R\bar{u} = \bar{u} \cdot \bar{u} = \bar{v} \cdot \bar{v}$$

$$\Downarrow$$

$$R\bar{v} \cdot R\bar{v} = \bar{v} \cdot \bar{v} = \bar{u} \cdot \bar{u}$$

$$R\bar{u} \cdot R\bar{v} = -\sin\theta \cos\theta \bar{u} \cdot \bar{u} + \cos^2\theta \bar{u} \cdot \bar{v} - \sin^2\theta \bar{v} \cdot \bar{u} \\ + \sin\theta \cos\theta \bar{v} \cdot \bar{v} = 0$$

It is convenient to choose an orthonormal basis

Orthogonal basis

$$R e_1 = \cos \vartheta e_1 + \sin \vartheta e_2$$

$$R e_2 = -\sin \vartheta e_1 + \cos \vartheta e_2$$

matrix of R

$$\begin{pmatrix} \cos \vartheta & -\sin \vartheta \\ \sin \vartheta & \cos \vartheta \end{pmatrix}$$

we arrange components in columns

$$R e_1 \cdot e_1 = \cos \vartheta$$

$$R e_1 \cdot e_2 = \sin \vartheta$$

...

$$R^T e_1 \cdot e_1 = e_1 \cdot R e_1 = R e_1 \cdot e_1 = \cos \vartheta$$

$$R^T e_1 \cdot e_2 = R e_2 \cdot e_1 = -\sin \vartheta$$

matrix of R^T

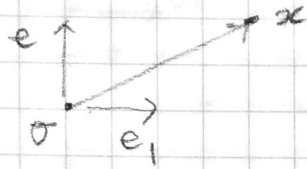
In general

$$A e_1 = a_{11} e_1 + a_{21} e_2$$

$$A e_2 = a_{12} e_1 + a_{22} e_2$$

$$[A] = \begin{pmatrix} a_{11} & a_{12} \\ a_{21} & a_{22} \end{pmatrix}$$

Coordinates (cartesian)



$$(x - \sigma) = x_1 e_1 + x_2 e_2$$

$$P_A = P_0 + R(\bar{P}_A - \bar{P}_0)$$

$$\begin{pmatrix} x_{1A} \\ x_{2A} \end{pmatrix} = \begin{pmatrix} x_{10} \\ x_{20} \end{pmatrix} + \begin{pmatrix} \cos \theta & -\sin \theta \\ \sin \theta & \cos \theta \end{pmatrix} \begin{pmatrix} \bar{x}_{1A} - \bar{x}_{10} \\ \bar{x}_{2A} - \bar{x}_{20} \end{pmatrix}$$

extending rotations to 3D

$$R a = \lambda a \quad \|R a\| = \|a\|$$

$$\Rightarrow |\lambda| = 1 \quad \lambda = \alpha \pm \beta i \quad \lambda = \pm 1$$

[2014-03-04] Tuesday (9-10)₂

9:00-11:00 (9:00-10:45, no breaks)

$$Ra = \lambda a$$

$$Ra \cdot Ra = a \cdot a \Rightarrow \lambda^2 = 1$$

extending V to the complex field \mathbb{C}

$$u \cdot v = \overline{v} \cdot u$$

$$(\alpha u) \cdot v = \alpha (u \cdot v)$$

$$\Rightarrow u \cdot (\beta v) = \overline{\beta v} \cdot u = \overline{\beta} \overline{v} \cdot u = \overline{\beta} u \cdot v$$

$$Ra \cdot Ra = \lambda \overline{\lambda} = |\lambda|^2$$

$$\lambda_{1,2} = \alpha \pm i\beta \quad \alpha^2 + \beta^2 = 1 \quad \lambda_3 = \pm 1$$

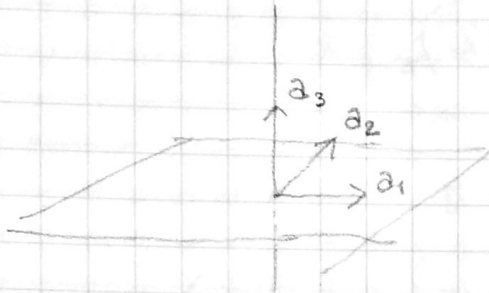


$$\alpha = \cos \theta$$

$$\beta = \sin \theta$$

$$\begin{pmatrix} \cos \theta + i \sin \theta & 0 & 0 \\ 0 & \cos \theta - i \sin \theta & 0 \\ 0 & 0 & 1 \end{pmatrix}$$

$$\begin{pmatrix} \cos \theta & -i \sin \theta & 0 \\ i \sin \theta & \cos \theta & 0 \\ 0 & 0 & 1 \end{pmatrix}$$



orthogonal to each other

$$R a_3 = a_3$$

$$R a_1 \cdot a_3 = 0$$

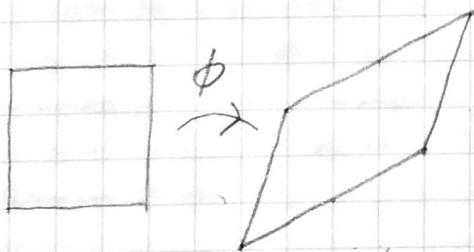
$$R a_2 \cdot a_3 = 0$$

We generalize rigid deformations to affine deformations

$$\phi(\bar{P}_A) = \phi(\bar{P}_O) + F(\bar{P}_A - \bar{P}_O)$$

where F is a non singular tensor ($F(\bar{u}) = 0 \Leftrightarrow \bar{u} = 0$)

- Straight lines are deformed into straight lines
- parallel segments are deformed into parallel segments
- a square is deformed into a parallelogram
- a cube is deformed into a parallelepiped
- a circle is deformed into an ellipse
- a sphere is deformed into an ellipsoid

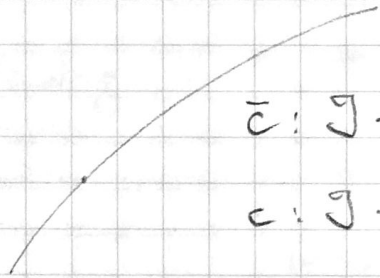


An affine deformation is also called a homogeneous deformation

[2014-03-05] Wednesday (11-12)₂

11:00 - 13:00 A1,3

Tangent vectors



$\bar{c}: \mathcal{J} \rightarrow \mathcal{E}$ curve on $\bar{\mathcal{R}} = \text{im } \bar{p}$

$c: \mathcal{J} \rightarrow \mathcal{E}$ curve on $\mathcal{R} = \text{im } p$

Parameterization of the reference shape

$$\kappa(s_1, s_2, s_3) \in \bar{\mathcal{R}}$$

in general it is a piecewise parameterization

so κ^{-1} does exist locally in general

$$\kappa: \mathcal{D} \rightarrow \bar{\mathcal{R}} \subset \mathcal{E}$$

↑ parameterization domain

(s_1, s_2, s_3) are also called "material coordinates"

$$\phi: \bar{\mathcal{R}} \rightarrow \mathcal{R} \subset \mathcal{E}$$

↑ \bar{p}
 \mathcal{B}

$$\phi: \bar{\mathcal{R}} \rightarrow \mathcal{R}$$

$$\uparrow \kappa$$

$$\mathcal{D}$$

$$\phi_{\kappa} = \phi \circ \kappa: \mathcal{D} \rightarrow \mathcal{R}$$

$$\phi(\kappa(s_1, s_2, s_3)) \in \mathcal{R}$$

parameterization
of the current shape
(convected coordinates)

By using a coordinate system

$$\begin{aligned} \phi_{\kappa}(s_1, s_2, s_3) &= \sigma + \phi_{\kappa 1}(s_1, s_2, s_3) e_1 \\ &= \phi(\kappa(s_1, s_2, s_3)) = \quad + \phi_{\kappa 2}(s_1, s_2, s_3) e_2 \\ &\quad + \phi_{\kappa 3}(s_1, s_2, s_3) e_3 \end{aligned}$$

$$\bar{c}_1(h) = \bar{p}_0 + h e_1 = \kappa(s_1, s_2) + h e_1$$

$$\underbrace{\quad}_{x} \in \bar{\mathcal{R}}$$

let us define κ , for example, by using coordinates in \mathcal{D}

$$\kappa(s_1, s_2, s_3) = \sigma + s_1 e_1 + s_2 e_2 + s_3 e_3$$

Thus

$$\bar{c}_1(h) = \sigma + s_1 e_1 + \dots + h e_1 = \kappa(s_1 + h, s_2, s_3)$$

$$c_1(h) = \phi(\bar{c}_1(h)) = \phi(\kappa(s_1+h, s_2, s_3))$$

$$c_1(h) = \phi_{\kappa}(s_1+h, s_2, s_3)$$

$$\begin{aligned} c_1(h) &= \sigma + \phi_{\kappa_1}(s_1+h, s_2, s_3) e_1 \\ &\quad + \phi_{\kappa_2}(s_1+h, s_2, s_3) e_2 \\ &\quad + \phi_{\kappa_3}(s_1+h, s_2, s_3) e_3 \end{aligned}$$

$$c_1'(0) = \lim_{h \rightarrow 0} \frac{1}{h} (c_1(h) - c_1(0)) = \dots$$

$$= \partial_1 \phi_{\kappa_1} e_1 + \partial_1 \phi_{\kappa_2} e_2 + \partial_1 \phi_{\kappa_3} e_3$$

[...]

$$c_2'(0) = \dots$$

$$c_3'(0) = \dots$$

$$\bar{c}(h) = \kappa(s_1+h\alpha_1, s_2+h\alpha_2, s_3+h\alpha_3)$$

$$\begin{aligned} c(h) &= \sigma + \phi_{\kappa_1}(s_1+h\alpha_1, s_2+h\alpha_2, s_3+h\alpha_3) e_1 \\ &\quad + \dots \end{aligned}$$

⇒ there exists a linear transformation (tensor)

$F: \mathcal{V} \rightarrow \mathcal{V}$, called the deformation gradient, such that

$$F \bar{c}'_1 = c'_1, \quad F \bar{c}'_2 = c'_2, \quad F(\alpha_1 \bar{c}'_1 + \alpha_2 \bar{c}'_2) = \alpha_1 c'_1 + \alpha_2 c'_2$$

(13-14)₃ Monday [2014-03-10]

15:00 → 18:00

A13

Local deformation

$$c_1(h) = \phi(\bar{p}_0) + F(\bar{p}_0) (\bar{c}_1(h) - \bar{c}_1(0)) + o(h)$$

$$o(h) := c_1(h) - \underbrace{\left(\phi(\bar{p}_0) + F(\bar{p}_0) (\bar{c}_1(h) - \bar{c}_1(0)) \right)}_{c_1(0)}$$

$$\lim_{h \rightarrow 0} o(h) = 0$$

$$\lim_{h \rightarrow 0} \frac{c_1(h) - c_1(0)}{h} - F(\bar{p}_0) \bar{c}'_1(0) = 0$$

$$\lim_{h \rightarrow 0} \frac{o(h)}{h} = c'_1(0) - F(\bar{p}_0) \bar{c}'_1(0) = 0$$

$o(h) \rightarrow 0$ faster than h

That is why we study AFFINE DEFORMATIONS

Polar decomposition of the deformation gradient

$$F = RU \quad R^T R = I \quad \det R = 1$$

$$U \in P_{\text{sym}}$$

$$C := F^T F = (RU)^T (RU) = U^2$$

$$U = \sqrt{F^T F}$$

$$[C] = [A] \text{diag}(\eta_1, \eta_2, \eta_3) [A]^T$$

$$[u_1, u_2, u_3]^T [u_i] = \begin{pmatrix} 1 \\ 0 \\ 0 \end{pmatrix}$$

$$[u_1, u_2, u_3] \text{diag}(\eta_1, \eta_2, \eta_3) \begin{pmatrix} 1 \\ 0 \\ 0 \end{pmatrix} = [u_1] \eta_1$$

$$C = \eta_1 u_1 \otimes u_1 + \eta_2 u_2 \otimes u_2 + \eta_3 u_3 \otimes u_3$$

$$C u_1 = \eta_1 u_1 \quad \dots$$

(15-16)₃ Tuesday [2014-03-11]
A1.3 09:00-11:00

$$\text{vol}(u_1, u_2, u_3) \quad \text{vol} : \mathbb{V} \times \mathbb{V} \times \mathbb{V} \rightarrow \mathbb{R}$$

$$\begin{cases} \text{vol}(u_1 + v, u_2, u_3) = \text{vol}(u_1, u_2, u_3) + \text{vol}(v, u_2, u_3) \\ \text{vol}(\alpha u_1, u_2, u_3) = \alpha \text{vol}(u_1, u_2, u_3) \\ \text{vol}(u_2, u_1, u_3) = -\text{vol}(u_1, u_2, u_3) \end{cases}$$

$$\Rightarrow \text{vol}(u_1, u_1, u_3) = -\text{vol}(u_1, u_1, u_3) = 0$$

$$\text{vol}(\underbrace{u_1 - u_1}_0, u_2, u_3) = \text{vol}(u_1, u_2, u_3) - \text{vol}(u_1, u_2, u_3) = 0$$

$$\text{vol}(\alpha_2 u_2 + \alpha_3 u_3, u_2, u_3) = 0$$

$$\text{vol}(u_1, u_2, u_3) = 0 \quad \text{for } u_1, u_2, u_3 \text{ linearly independent}$$

$$\Rightarrow \text{vol} = 0$$

$$A_{g_1} := (u_1, u_2, u_3)$$

$$V = \text{vol}(u_1, u_2, u_3) = \text{vol}(\underbrace{w_1 + (u_1 \cdot n_1) n_1}_{w_1}, u_2, u_3)$$

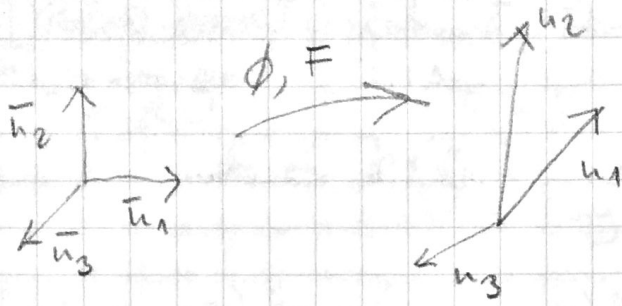
$$w_1 := u_1 - (u_1 \cdot n_1) n_1$$

$$n_1 \text{ unit normal vector} \quad \Rightarrow \quad w_1 \cdot n_1 = u_1 \cdot n_1 - (u_1 \cdot n_1)(n_1 \cdot n_1) = 0$$

$$w_1 = \alpha_2 u_2 + \alpha_3 u_3$$

$$\Rightarrow V = h_1 A_{g_1} \quad h_1 := (u_1 \cdot n_1)$$

[Notebook page scanned on 2014-06-20]



How volumes and areas are changed by F ?

$$V = \text{vol}(u_1, u_2, u_3) = \text{vol}(Fu_1, Fu_2, Fu_3)$$

$$\frac{V}{\bar{V}} = \frac{\text{vol}(Fu_1, Fu_2, Fu_3)}{\text{vol}(\bar{u}_1, \bar{u}_2, \bar{u}_3)}$$

Choose any basis $\{e_1, e_2, e_3\}$

For example $e_1 = \bar{u}_1, e_2 = \bar{u}_2, e_3 = \bar{u}_3$

In the very simple case of planar deformation

$$Fe_1 = f_{11}e_1 + f_{21}e_2 + 0e_3$$

$$Fe_2 = f_{12}e_1 + f_{22}e_2 + 0e_3$$

$$Fe_3 = e_3$$

we get

$$\frac{\text{vol}(Fe_1, Fe_2, Fe_3)}{\text{vol}(e_1, e_2, e_3)} = f_{11}f_{22} - f_{21}f_{12}$$

The ratio does not depend on the basis ⁽¹⁾

It does not depend on the volume fraction

That is why we define

$$\det F = \frac{\text{vol}(F\bar{u}_1, F\bar{u}_2, F\bar{u}_3)}{\text{vol}(\bar{u}_1, \bar{u}_2, \bar{u}_3)}$$

⁽¹⁾ just because the definition of vol does not rely on any basis

Wednesday [2014-03-12] 11:00-13:00 A1.3 (17-18)₃

$$\frac{V}{\bar{V}} = \det F$$

let us consider

$$V = h_1 \Delta_{\mathcal{F}_1} \quad h_1 := u_1 \cdot n_1$$

$$\bar{V} = \bar{h}_1 \bar{\Delta}_{\bar{\mathcal{F}}_1} \quad \bar{h}_1 := \bar{u}_1 \cdot \bar{n}_1$$

$$\frac{\Delta_{\mathcal{F}_1}}{\bar{\Delta}_{\bar{\mathcal{F}}_1}} = \frac{V}{\bar{V}} \frac{\bar{h}_1}{h_1} = \frac{\bar{h}_1}{h_1} \det F$$

$$h_1 := u_1 \cdot n_1 \quad u_2 \cdot n_1 = 0 \quad u_3 \cdot n_1 = 0$$

$$h_1 = F \bar{u}_1 \cdot n_1 \quad F \bar{u}_2 \cdot n_1 = 0 \quad F \bar{u}_3 \cdot n_1 = 0$$

$F \bar{u}_2$ and $F \bar{u}_3$ are the edges of \mathcal{F}_1

$$h_1 = \bar{u}_1 \cdot F^T n_1 \quad \bar{u}_2 \cdot F^T n_1 = 0 \quad \bar{u}_3 \cdot F^T n_1 = 0$$

$\Rightarrow F^T n_1$ is orthogonal to

\bar{u}_2 and \bar{u}_3 which are the edges of $\bar{\mathcal{F}}_1$

Hence F^T pulls back n_1 to a vector normal to $\bar{\mathcal{F}}_1$

$$F^T n_1 = k_1^{-1} \bar{m}_1$$

$$n_1 = k_1^{-1} F^{-T} \bar{m}_1 \quad \Rightarrow \quad h_1 = \bar{u}_1 \cdot F^T n_1 = \bar{u}_1 \cdot \bar{m}_1 k_1^{-1}$$

$$h_1 = \bar{h}_1 k_1^{-1}$$

$$k_1 = \frac{\bar{h}_1}{h_1} \quad \Leftarrow$$

$$F^{-T} \bar{m}_1 = k_1 n_1 = \frac{A_{g_1}}{A_{\bar{g}_1}} \frac{1}{\det F} n_1$$

$$\underbrace{(\det F)}_{\text{cof } F} F^{-T} \bar{m}_1 = \frac{A_{g_1}}{A_{\bar{g}_1}} n_1$$

Matrix of cof F

$\text{vol}(e_1, e_2, e_3) = 1$ orthonormal basis

$$(\text{cof } F) e_1 = \frac{A_{g_1} n_1}{A_{\bar{g}_1} \rightarrow 1} = A_{g_1} n_1$$

$$(\text{cof } F) e_1 \cdot e_i = A_{g_1} n_1 \cdot e_i = (n_1 \cdot e_i) \text{vol}(n_1, F e_2, F e_3)$$

$$e_i = w_{ii} + (e_i \cdot n_1) n_1 \quad \Rightarrow \quad w_{ii} \cdot n_1 = 0$$

$$\begin{aligned} (\text{cof } F) e_1 \cdot e_i &= \text{vol}((n_1 \cdot e_i) n_1, F e_2, F e_3) \\ &= \text{vol}(e_i - w_{ii}, F e_2, F e_3) \\ &= \text{vol}(e_i, F e_2, F e_3) \end{aligned}$$

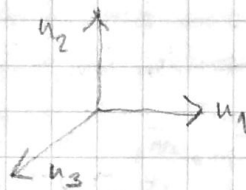
$$\underbrace{(\text{cof } F) e_2 \cdot e_i}_{= \frac{A_{32}}{\bar{A}_{32}} n_2} = A_{32} n_2 \cdot e_i \quad \bar{A}_{32} = 1$$

$$(\text{cof } F) e_2 \cdot e_i = A_{32} (n_2 \cdot e_i) = (n_2 \cdot e_i) \text{vol}(Fe_1, n_2, Fe_3)$$

$$A_{32} = \text{vol}(e_1, n_2, e_3)$$

$$e_i = n_{2i} + (e_i \cdot n_2) n_2$$

$$\Rightarrow n_{2i} \cdot n_2 = 0$$



$$\begin{aligned} (\text{cof } F) e_2 \cdot e_i &= \text{vol}(Fe_1, e_i - n_{2i}, Fe_3) \\ &= \text{vol}(Fe_1, e_i, Fe_3) \end{aligned}$$

recall that $\text{vol}(u_1, u_2, u_3)$ is equal to the det of the matrix made up with the vectors components

$$= \det \begin{pmatrix} f_{11} & \cdot & f_{13} \\ f_{21} & \cdot & f_{23} \\ f_{31} & \cdot & f_{33} \end{pmatrix}$$

$$= Fe_1 \cdot e_i \times Fe_3 = -e_i \cdot Fe_1 \times Fe_3 \quad [111]$$

$$\text{tr } A = \frac{\text{vol}(Ae_1, Ae_2, Ae_3)}{\text{vol}(e_1, e_2, e_3)} + \dots$$

$$\begin{aligned} \frac{d}{dt} \det F(t) &= \frac{\text{vol}(\dot{F}\bar{u}_1, \dot{F}\bar{u}_2, \dot{F}\bar{u}_3)}{\text{vol}(\bar{u}_1, \bar{u}_2, \bar{u}_3)} + \dots \\ &= \frac{\text{vol}(\dot{F}F^{-1}u_1, u_2, u_3)}{\text{vol}(\bar{u}_1, \bar{u}_2, \bar{u}_3)} + \dots \\ &= \frac{\text{vol}(\dot{F}F^{-1}u_1, u_2, u_3)}{\text{vol}(\bar{u}_1, \bar{u}_2, \bar{u}_3)} \frac{\text{vol}(u_1, u_2, u_3)}{\text{vol}(u_1, u_2, u_3)} + \dots \\ &= \frac{\text{vol}(\dot{F}F^{-1}u_1, u_2, u_3)}{\text{vol}(u_1, u_2, u_3)} \det F + \dots \\ &= \det F \text{tr}(\dot{F}F^{-1}) \end{aligned}$$

$$\det AB = \frac{\text{vol}(ABe_1, ABe_2, ABe_3)}{\text{vol}(e_1, e_2, e_3)}$$

$$\det B = \frac{\text{vol}(Be_1, Be_2, Be_3)}{\text{vol}(e_1, e_2, e_3)}$$

$$\det A = \frac{\text{vol}(ABe_1, ABe_2, ABe_3)}{\text{vol}(Be_1, Be_2, Be_3)}$$

$$\Rightarrow \det A \det B = \det AB$$

$$F^T m_1 = k_1^{-1} \bar{m}_1 = \frac{h_1}{\bar{h}_1} \bar{m}_1 \quad h_1 = u_1 \cdot n_1; \quad \bar{h}_1 = \bar{u}_1 \cdot \bar{n}_1$$

There are 2 couples of dual bases

$$\left\{ \begin{array}{l} F^T m_i = a_{1i} u_1 + a_{2i} u_2 + a_{3i} u_3 \\ m_i = b_{1i} \bar{u}_1 + b_{2i} \bar{u}_2 + b_{3i} \bar{u}_3 \end{array} \right. \quad \begin{array}{l} \{u_i\}, \{m_j\} \\ \{\bar{u}_i\}, \{\bar{m}_j\} \end{array} \quad \begin{array}{l} (*) \\ (0) \end{array}$$

$$\left\{ \begin{array}{l} F^T m_1 \cdot m_2 = a_{21} u_2 \cdot m_2 = a_{21} h_2; \quad F^T m_1 \cdot m_1 = a_{11} h_1 \quad (*) \\ m_2 \cdot \bar{m}_1 = b_{12} \bar{u}_1 \cdot \bar{m}_1 = b_{12} \bar{h}_1; \quad m_1 \cdot \bar{m}_2 = b_{11} \bar{h}_1 \quad (*) \end{array} \right. \quad \begin{array}{l} (1) \\ (2) \end{array}$$

$$\left. \begin{array}{l} F^T m_1 \cdot \bar{u}_2 = m_1 \cdot u_2 = 0; \quad F^T m_1 \cdot \bar{u}_3 = 0 \\ F^T m_2 \cdot \bar{u}_1 = 0; \quad F^T m_2 \cdot \bar{u}_3 = 0 \\ F^T m_3 \cdot \bar{u}_1 = 0; \quad F^T m_3 \cdot \bar{u}_2 = 0 \end{array} \right\} \Rightarrow \begin{array}{l} F^T m_1 = k_1^{-1} \bar{m}_1 \\ F^T m_2 = k_2^{-1} \bar{m}_2 \\ F^T m_3 = k_3^{-1} \bar{m}_3 \end{array} \quad (*) \quad (3)$$

$$\left\{ \begin{array}{l} F^T m_1 \cdot m_1 = k_1^{-1} \bar{m}_1 \cdot m_1 = a_{11} h_1 \Rightarrow \frac{h_1}{\bar{h}_1} b_{11} \bar{h}_1 = a_{11} h_1 \\ F^T m_1 \cdot m_2 = k_1^{-1} \bar{m}_1 \cdot m_2 = a_{21} h_2 \Rightarrow \frac{h_1}{\bar{h}_1} b_{12} \bar{h}_1 = a_{21} h_2 \end{array} \right. \quad (*) \quad (3)$$

$$\det F^T = \frac{\text{vol}(F^T m_1, F^T m_2, F^T m_3)}{\text{vol}(m_1, m_2, m_3)} = \frac{\langle \det a_{ij} \rangle \text{vol}(u_1, u_2, u_3)}{\langle \det b_{ij} \rangle \text{vol}(\bar{u}_1, \bar{u}_2, \bar{u}_3)} = \det F$$

$$\begin{pmatrix} h_1 \\ h_2 \\ h_3 \end{pmatrix} \begin{pmatrix} b_{11} & \dots \\ b_{21} & \dots \\ b_{31} & \dots \end{pmatrix} \begin{pmatrix} h_1 \\ h_2 \\ h_3 \end{pmatrix}^{-1} = \begin{pmatrix} a_{11} & \dots \\ a_{21} & \dots \\ a_{31} & \dots \end{pmatrix}^T \quad h_i b_{ij} = a_{ji} h_j \quad (*)$$

It is much easier to rely on the determinant's independence of any basis, choose an orthonormal basis and exploit the property of the matrices of being the transpose of each other thus resulting in the same expression for the determinant.

(19-20)₄ Monday [2014-03-17] 16:00-18:00
A1.3

Velocity fields (vector fields)

$$v: \mathcal{R} \subset \mathcal{E} \rightarrow \mathcal{V}$$

velocity gradient $\uparrow \phi$

$$\bar{v}: \bar{\mathcal{R}} \rightarrow \mathcal{V}$$

$\uparrow \kappa$
 \mathcal{D}

$$c(h) = \sigma + c_1(h)e_1 + c_2(h)e_2 + c_3(h)e_3$$

$$c'(0) = \lim_{h \rightarrow 0} \frac{1}{h} (c(h) - c(0)) = c_1'(0)e_1 + c_2'(0)e_2 + c_3'(0)e_3$$

scalar functions

$$r(c(h)) = r_1(c_1(h), c_2(h), c_3(h))e_1 + \dots$$

$$\lim_{h \rightarrow 0} \frac{1}{h} (r(c(h)) - r(c(0))) =$$

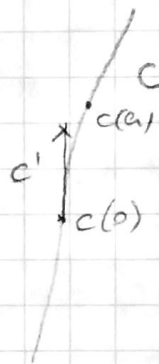
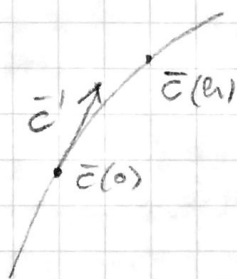
$$\begin{aligned} & \partial_1 r_1 c_1' e_1 + \partial_2 r_1 c_2' e_1 + \partial_3 r_1 c_3' e_1 \\ & + \partial_1 r_2 c_1' e_2 + \partial_2 r_2 c_2' e_2 + \partial_3 r_2 c_3' e_2 \\ & + \partial_1 r_3 c_1' e_3 + \partial_2 r_3 c_2' e_3 + \partial_3 r_3 c_3' e_3 \end{aligned}$$

$$= \nabla r c'$$

[Notebook page scanned on 2014-06-20]

$$\begin{pmatrix} \partial_1 \tau_1 & \partial_2 \tau_1 & \partial_3 \tau_1 \end{pmatrix} \begin{pmatrix} c_1' \\ c_2' \\ c_3' \end{pmatrix}$$

$$\uparrow \\ [\nabla \tau]$$



PULL BACK OF VECTOR FIELDS

$$\tilde{\tau}(c(h)) = \tau(c(h))$$

$$\frac{\tilde{\tau}(c(h)) - \tilde{\tau}(c(0))}{h} = \frac{\tau(c(h)) - \tau(c(0))}{h}$$

taking the limit

$$\nabla \tilde{\tau} c' = \nabla \tau c'$$

$$\nabla \tilde{\tau} c' = \nabla \tau F c'$$

$$\Rightarrow \nabla \tilde{\tau} = \nabla \tau F$$

$$\tau(c(h)) = \tau(c(0)) + \nabla \tau (c(h) - c(0)) + o(h)$$

$$\frac{\tau(c(h)) - \tau(c(0))}{h} = \nabla \tau \frac{c(h) - c(0)}{h} + \frac{o(h)}{h}$$

$$\Rightarrow \lim_{h \rightarrow 0} \frac{o(h)}{h} = 0$$

[Notebook page scanned on 2014-06-20]

(21-22) Tuesday [2014-03-18] 9:00-11:00
A1.3

velocity field

$$v(c(t)) = v(c(0)) + \overset{L}{\nabla v} (c(t) - c(0)) + o(t)$$

$$c(t) = c(0) + F (\bar{c}(t) - \bar{c}(0)) + o(t)$$

$$L := \nabla v; \quad F := \nabla \phi$$

let us differentiate w.r.t. time

$$\dot{c}(t) = \dot{c}(0) + \dot{F} (\bar{c}(t) - \bar{c}(0)) + o(t)$$

$$F^{-1} (c(t) - c(0)) = (\bar{c}(t) - \bar{c}(0)) + o(t)$$

$$\dot{c}(t) = \dot{c}(0) + \dot{F} F^{-1} (c(t) - c(0)) + o(t)$$

$$\Rightarrow \nabla v = \dot{F} F^{-1}$$

$$\nabla \bar{v} = \nabla v F = \dot{F}$$

we assumed $v(c(t)) = \dot{c}(t) \mid v(c(t), t) = \dot{c}(t, t)$

Differentiating again

$$\frac{d}{dt} v(c(t), t) = \overset{\substack{\uparrow \\ \text{fixed trajectory}}}{\nabla v} \dot{c} + \overset{\substack{\downarrow \\ \text{fixed position}}}{\frac{\partial}{\partial t}} v(x, t)$$

Acceleration from spatial velocity field

$$\bar{w}(\bar{c}(R)) = w(c(R))$$

moving $c(R)$, fixed w

$$\bar{w}(\bar{c}(R)) = w(c(R, t))$$

$$\downarrow \phi(\bar{c}(R), t) \nearrow$$

velocity field $v(x, t)$

$$\bar{v}(\bar{c}(R), t) = v(c(R, t), t)$$

acceleration field

$$\bar{a}(\bar{c}(R), t) = \frac{d}{dt} \bar{v}(\bar{c}(R), t)$$

$$= \frac{d}{dt} v(c(R, t), t)$$

pull-back of
any vector field w

velocity field

$$\bar{v}(\bar{P}_A, t) = \frac{d}{dt} P_A(t)$$

$$= \dot{P}_A(t) = v(P_A(t), t)$$

acceleration field

$$\bar{a}(\bar{P}_A, t) = \frac{d}{dt} \bar{v}(\bar{P}_A, t)$$

$$= \frac{d}{dt} v(P_A(t), t)$$

\bar{v}, \bar{a} referential description (Lagrangian)

v, a spatial description (Eulerian)

Time differentiation of the velocity field

$$\frac{d}{dt} \bar{v}(\bar{c}(R), t) = \frac{d}{dt} v(c(R, t), t)$$

$$= \nabla v \dot{c} + \frac{\partial}{\partial t} v$$

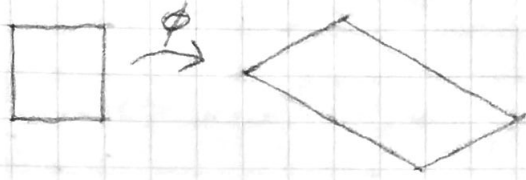
v field at time t \nearrow $\frac{\partial}{\partial t} v$ at a fixed position $c(R, t)$
 $v(c(R, t), t)$

$$\bar{a}(\bar{c}(R), t) = a(c(R, t), t) = (\nabla v) v + \frac{\partial}{\partial t} v$$

$$a(x, t) = (\nabla v(x, t)) v(x, t) + \frac{\partial}{\partial t} v(x, t)$$

(23-24)₄ Wednesday [2014-03-13]

Integrals, Jacobian, total volume, mass densities



ϕ

$$\rho_0 \bar{V} \stackrel{\text{conservation of mass}}{=} \rho V = (\rho \det F) \bar{V} \quad \begin{array}{l} \text{initial value} \\ \downarrow \\ \rho_0 = \rho \det F \\ \text{constant quantity} \end{array}$$

$$\frac{d}{dt} \rho_0 = 0 \Rightarrow 0 = \dot{\rho} + \rho \operatorname{div} v$$

$$\int_{\mathcal{R}} \rho dV = \int_{\bar{\mathcal{R}}} \rho \det F dV$$

possibly changing in time

mass conservation

$$\int_{\mathcal{R}} \rho_0 dV = \int_{\bar{\mathcal{R}}} \rho dV$$

isochoric deformation $\det F = 1$

$$\nabla v = \dot{F} F^{-1}$$

$$\frac{d}{dt} (\det F) = (\det F) \operatorname{tr} \dot{F} F^{-1} = 0 \Rightarrow \operatorname{tr} \nabla v = 0$$

$\operatorname{div} v = \operatorname{tr} \nabla v$ $\operatorname{div} v = 0$

$$\int_{\bar{\mathcal{R}}} dV = \operatorname{vol}(\bar{\mathcal{R}}); \quad \int_{\mathcal{R}} dV = \operatorname{vol}(\mathcal{R})$$

$$\int_{\mathcal{R}} dV = \int_{\bar{\mathcal{R}}} \det F dV$$

$$\frac{d}{dt} \int_{\mathcal{R}} dV = \frac{d}{dt} \int_{\bar{\mathcal{R}}} \det F dV = \int_{\bar{\mathcal{R}}} \frac{d}{dt} \det F dV = \int_{\bar{\mathcal{R}}} (\det F) \operatorname{tr} \nabla v dV$$

$$= \int_{\mathcal{R}} \operatorname{tr} \nabla v dV = \int_{\mathcal{R}} \operatorname{div} v dV = \int_{\partial \mathcal{R}} v \cdot n dA$$

[Notebook page scanned on 2014-06-20]

$$\nabla \mathbf{r} = \dot{\mathbf{F}} \mathbf{F}^{-1}$$

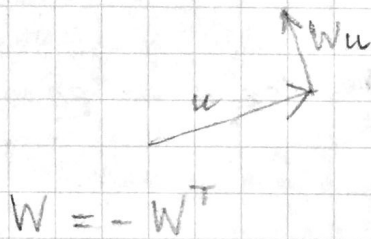
$$\nabla \mathbf{r} = \dot{\mathbf{R}} \mathbf{R}^T \in \text{Skw}$$

$$\frac{d}{dt} \mathbf{R} \mathbf{R}^T = \dot{\mathbf{R}} \mathbf{R}^T + \mathbf{R} \dot{\mathbf{R}}^T = 0$$

SPIN $\mathbf{W} := \dot{\mathbf{R}} \mathbf{R}^T$

$$\mathbf{W} \mathbf{u} \cdot \mathbf{u} = 0$$

$$\mathbf{W} \mathbf{u} \cdot \mathbf{u} = -\mathbf{u} \cdot \mathbf{W} \mathbf{u}$$



$$\mathbf{W} \mathbf{a} = \lambda \mathbf{a}$$

$$\mathbf{W} \mathbf{a} \cdot \mathbf{a} = 0 \Rightarrow \lambda (\mathbf{a} \cdot \mathbf{a}) = 0$$

$$\Downarrow$$

$$\lambda = 0$$

$$\mathbf{D} \in \text{Sym}$$

STRETCHING



(25-26)₅ Monday [2014-03-24] A1.3
16:00-18:00

$$W a = \lambda a \Rightarrow \lambda = 0$$

$$\omega = a / \|a\|$$

$$W \omega = 0 \quad \text{vol}(w, u, r) = \omega \times u \cdot r \quad \forall r$$

$\text{vol}(e_1, e_2, e_3) = 1$

$$\text{vol}(w, u, r) = (\ast w) u \cdot r \quad \forall r, \forall u, \forall r$$

↑ Hodge star

$$\text{vol}(w, u, r) = W u \cdot r$$

Power as a linear functional on velocity fields

Force $\mathcal{F}(v)$ representation

special form
$$\mathcal{F}(v) = f_a \cdot v(p_a) + f_b \cdot v(p_b) + f_c \cdot v(p_c)$$

more general
$$\mathcal{F}(v) = \int_{\mathcal{R}} b(x) \cdot v(x) dV + \int_{\partial \mathcal{R}} t(x) \cdot v(x) dA$$

velocity field

$$v(c(t)) = v(c(0)) + \nabla v(c(0)) (r(c(t)) - r(c(0))) + o(t)$$

$$v(x) = v_0 + \nabla v (x - p_0) \quad \text{affine}$$

$$\nabla v = \dot{R} R^T = W \quad \text{rigid}$$

$$W u_A \cdot f_A = w \times u_A \cdot f_A$$

$$= f_A \cdot w \times (P_A - P_0) = (P_A - P_0) \times f_A \cdot w$$

$$\begin{aligned} \mathcal{F}(w) &= f_A \cdot r(P_A) + f_B \cdot r(P_B) + f_C \cdot r(P_C) \\ &= f_A \cdot r_0 + f_A \cdot w \times (P_A - P_0) + \dots \\ &= f_A \cdot r_0 + (P_A - P_0) \times f_A \cdot w + \dots \\ &= \dots = f \cdot r_0 + m \cdot w \end{aligned}$$

$$\mathcal{F}(w) = \int_{\mathcal{R}} b(x) \cdot r(x) dV + \int_{\partial \mathcal{R}} t(x) \cdot r(x) dA$$

Bulk force
(density with respect
to the volume)

per unit volume

Traction
(density with respect
to the area)

per unit area

$$\begin{aligned} &= \left(\int_{\mathcal{R}} b(x) dV + \int_{\partial \mathcal{R}} t(x) dA \right) \cdot r_0 \\ &+ \left(\int_{\mathcal{R}} (x - p_0) \times b(x) dV + \int_{\partial \mathcal{R}} (x - p_0) \times t(x) dA \right) \cdot w \\ &= f \cdot r_0 + m \cdot w \end{aligned}$$

(27-28)₅

Thursday [2014-03-25]

example 1

$$t(x) = 0, \quad b(x) = \rho g \quad g \text{ is a vector}$$

$$f = \int_{\mathcal{R}} \rho g \, dV = \rho g \, \text{Vol}(\mathcal{R}) = \underbrace{\rho \, \text{Vol}(\mathcal{R})}_{\text{total mass}} g$$

$$m = \int_{\mathcal{R}} (x - p_0) \times (\rho g) \, dV = \int_{\mathcal{R}} (x - p_0) \, dV \times (\rho g)$$

$$\frac{1}{V} \int_{\mathcal{R}} (x - p_0) \, dV = (x_G - p_0)$$

↑ barycenter of \mathcal{R}

if $p_0 = x_G$ then $m = 0$

example 2

$$b(x) = 0, \quad t(x) = \rho g \quad \rho \text{ "skin" mass density}$$

$$f = \int_{\partial\mathcal{R}} \rho g \, dA = \underbrace{\rho \, \text{Area}(\partial\mathcal{R})}_{\text{total mass}} g; \quad m = \int_{\partial\mathcal{R}} (x - p_0) \, dA \times (\rho g)$$

$$\frac{1}{A} \int_{\partial\mathcal{R}} (x - p_0) \, dA = (x_G - p_0)$$

↑ barycenter of $\partial\mathcal{R}$

Balance principle

$$\mathcal{F}(v) = 0 \quad \forall v$$

$$v(x) = v_0 + W(x - p_0)$$

$$\Rightarrow \mathcal{F}(v) = f \cdot v_0 + m \cdot \omega$$

$$\mathcal{F}(v) = 0 \quad \forall v \Leftrightarrow f = 0, m = 0$$

v general continuous and differentiable vector field

$$\mathcal{F}(v) = \int_{\mathcal{R}} b(x) \cdot v(x) dV + \int_{\partial \mathcal{R}} t(x) \cdot v(x) dV$$

$$\mathcal{F}(v) = 0 \quad \forall v \Rightarrow b = 0, t = 0 \quad \text{no force is allowed}$$

A convenient approach consists in setting

$$\mathcal{F}(v) = \overset{\text{ext}}{\mathcal{F}}(v) + \overset{\text{int}}{\mathcal{F}}(v)$$

based on a suitable characterization of the two different parts, and still requiring

$$\mathcal{F}(v) = 0 \quad \forall v$$

In order to have some insight into the way of characterizing the force distribution let us consider just the extension of rigid velocity fields to affine velocity fields:

$$v(x) = v_0 + L(x - p_0)$$

where L is the velocity gradient (a tensor)

$$f^{\text{ext}}(v) = f_A \cdot v_A + f_B \cdot v_B + f_C \cdot v_C$$

$$= f_A \cdot v_0 + f_A \cdot L(p_A - p_0) + \dots$$

$$= f_A \cdot v_0 + (p_A - p_0) \otimes f_A \cdot L + \dots$$

$$= (f_A + f_B + f_C) \cdot v_0$$

$$+ ((p_A - p_0) \otimes f_A + (p_B - p_0) \otimes f_B + (p_C - p_0) \otimes f_C) \cdot L$$

$$= f \cdot v_0 + M \cdot L$$

(29-30)₅ Wednesday [2014-03-26]

$$f_A \cdot L (P_A - P_0) = \underbrace{(P_A - P_0) \otimes f_A \cdot L}_{\text{Curlin}} = f_A \otimes (P_A - P_0)$$

Curlin
↓
⊗, ⊗

$$M \cdot L = \text{tr}(M^T L)$$

$$\text{tr}(AB) = \frac{\text{vol}(ABe_1, e_2, e_3) + \dots}{\text{vol}(e_1, e_2, e_3)}$$

$$Be_i = b_{1i}e_1 + b_{2i}e_2 + b_{3i}e_3$$

$$Ae_i = a_{1i}e_1 + a_{2i}e_2 + a_{3i}e_3$$

any basis

$$ABe_i = b_{1i}Ae_1 + b_{2i}Ae_2 + b_{3i}Ae_3$$

$$= b_{1i}(a_{11}e_1 + a_{21}e_2 + a_{31}e_3)$$

$$+ b_{2i}(a_{12}e_1 + a_{22}e_2 + a_{32}e_3)$$

$$+ b_{3i}(a_{13}e_1 + a_{23}e_2 + a_{33}e_3)$$

$$\text{vol}(ABe_1, e_2, e_3) = (b_{11}a_{11} + b_{21}a_{12} + b_{31}a_{13})$$

$$\text{vol}(e_1, ABe_2, e_3) = (b_{12}a_{21} + b_{22}a_{22} + b_{32}a_{23})$$

$$\text{vol}(e_1, e_2, ABe_3) = (b_{13}a_{31} + b_{23}a_{32} + b_{33}a_{33})$$

If we exchange A and B the sum above will be left unchanged. So $\text{tr}(AB) = \text{tr}(BA)$

$$M \cdot L = \text{tr}(M^T L)$$

$$\begin{aligned} \text{tr}(M^T L) &= m_{11}^T v_{11} + m_{21}^T v_{12} + m_{31}^T v_{13} \\ &+ \\ &+ \end{aligned}$$

By using an orthonormal basis we realize that
since $m_{21}^T = m_{12}$ then

$$\text{tr}(M^T L) = \sum \sum m_{ij} v_{ij}$$

$$A \cdot W = \text{tr}(A^T W) = \text{tr}(W^T A) = W \cdot A$$

$$\left. \begin{array}{l} A^T = A \\ W^T = -W \end{array} \right\} \Rightarrow \begin{array}{c} \downarrow \qquad \qquad \downarrow \\ = \text{tr}(AW) = -\text{tr}(WA) = -\text{tr}(AW) \end{array}$$

$$\Rightarrow A \cdot W = 0$$

Properties of "tr"

$$\text{tr}(A) = \text{tr}(A^T)$$

$$\text{tr}(AB) = \text{tr}(BA)$$

$$\begin{aligned}
 M \cdot L &= (\text{skw } M + \text{sym } M) \cdot (W + D) \\
 &= \text{skw } M \cdot W + \cancel{\text{skw } M \cdot D} \\
 &\quad + \cancel{\text{sym } M \cdot W} + \text{sym } M \cdot D
 \end{aligned}$$

$$\begin{aligned}
 \mathbb{J}^{\text{ext}}(v) &= f \cdot r_0 + M \cdot L \\
 &= f \cdot r_0 + \text{skw } M \cdot W + \text{sym } M \cdot D
 \end{aligned}$$

v affine velocity field

$$v(x) = r_0 + L(x - p_0)$$

v rigid velocity field ($L = W$)

$$\mathbb{J}^{\text{ext}}(v) = f \cdot r_0 + \underbrace{\text{skw } M \cdot W}_{m \cdot \omega}$$

(31-32) Monday [2014-03-31]
16:00-18:00 A1,3

$$\mathcal{F}(v) = \mathcal{F}^{\text{ext}}(v) + \mathcal{F}^{\text{int}}(v)$$

$$\mathcal{F}(v) = 0 \quad \text{balance principle}$$

$$r(x) = r_0 + L(x - p_0)$$

$$\mathcal{F}^{\text{ext}} = f \cdot r_0 + M \cdot L$$

examples \rightarrow

let us set $\mathcal{F}^{\text{int}}(v) = -(z \cdot r_0 + T \cdot L) V_x$ \leftarrow why V_x ?

and give the following characterization

$$z \cdot r_0 + T \cdot L = 0 \quad \forall \text{ rigid velocity field}$$

$$r(x) = r_0 + W(x - p_0)$$

$$\Rightarrow z \cdot r_0 + T \cdot W = 0 \quad \forall r_0 \neq W$$

\Downarrow

$$z = 0, \quad \text{skw} T = 0$$

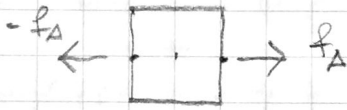
$$\mathcal{F}(v) = (f - z V_x) \cdot r_0 + (M - T V_x) \cdot L$$

$$= f \cdot r_0 + \text{skw} M \cdot W + (\text{sym} M - T V_x) \cdot D$$

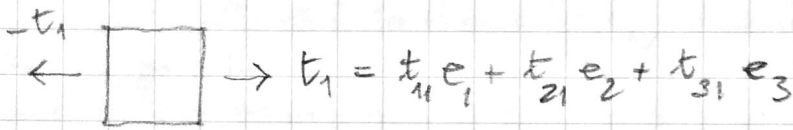
$$\mathcal{F}(v) = 0 \quad \forall v \Rightarrow f = 0; \text{skw} M = 0; \text{sym} M = T V_x$$

[Notebook page scanned on 2014-06-21]

$$M_A = \overset{\text{arm}}{(p_A - p_0)} \otimes \overset{\text{force}}{f_A} = \overset{\text{force}}{f_A} \otimes \overset{\text{arm}}{(p_A - p_0)} \quad (\text{Gurtin})$$



$$\frac{l_1}{2} e_1 \otimes (f e_1) = \frac{l_1}{2} f e_1 \otimes e_1$$



$$\frac{l_1}{2} e_1 \otimes t_1 \quad \int_{S_1} t \cdot n \, dA$$

$$e_1 \otimes e_2 = e_2 \otimes e_1$$

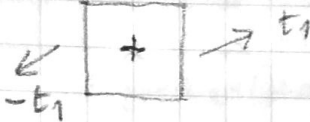
$$(e_1 \otimes e_2) e_1 = (e_1 \cdot e_1) e_2$$

$$[e_1 \otimes e_2] = \begin{pmatrix} 0 & 0 & 0 \\ 1 & 0 & 0 \\ 0 & 0 & 0 \end{pmatrix}$$

$$(e_2 \otimes e_1) e_1 = e_2 (e_1 \cdot e_1)$$

$$(e_1 \otimes t_1)_{n_1} = (e_1 \cdot n_1) t_1$$

$$\begin{aligned} M_A \cdot L &= (p_A - p_0) \otimes f_A \cdot L = f_A \cdot L (p_A - p_0) = f_A \cdot (\tilde{r}_A - \tilde{r}_0) \\ &= f_A \otimes (p_A - p_0) \cdot L = f_A \cdot L (p_A - p_0) = f_A \cdot (\tilde{w}_A - \tilde{w}_0) \end{aligned}$$



$$t_1 = t_{11} e_1 + t_{21} e_2 + t_{31} e_3$$

$$\int_{\mathcal{A}_1} t_i(x) \cdot r(x) dA = \int_{\mathcal{A}_1} t_i(x) \cdot r_0 dA + \int_{\mathcal{A}_1} t_i(x) \cdot L(x-p_0) dA$$

$$= \int_{\mathcal{A}_1} t_i dA \cdot r_0 + \int_{\mathcal{A}_1} (x-p_0) \otimes t_i dA \cdot L = \Delta_{\mathcal{A}_1} (g_{\mathcal{A}_1} - p_0) \otimes t_i \cdot L$$

useful parameterization

$$\left\{ \begin{array}{l} x-p_0 = \frac{l_1}{2} e_1 + s_2 e_2 + s_3 e_3 \\ -\frac{l_1}{2} \leq s_2 \leq \frac{l_2}{2} \quad -\frac{l_3}{2} \leq s_3 \leq \frac{l_3}{2} \end{array} \right.$$

$$g_{\mathcal{A}_1} - p_0 = \frac{1}{A_{\mathcal{A}_1}} \int_{\mathcal{A}_1} (x-p_0) dA$$

$$\int_{\mathcal{A}_1 \cup \mathcal{A}_1} t(x) \cdot r(x) dA = \int_{\mathcal{A}_1} t_i \cdot r(x) dA - \int_{\mathcal{A}_1} t_i \cdot r(x) dA$$

$$= (t_i A_{\mathcal{A}_1}) \otimes (g_{\mathcal{A}_1} - g_{\mathcal{A}_1}) \cdot L$$

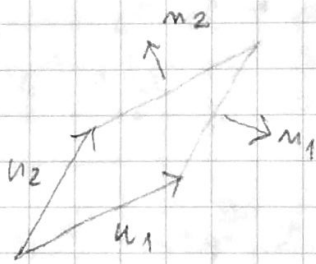
$$= (t_i l_2 l_3) \otimes (l_1 e_1) \cdot L = (t_i \otimes e_1 \cdot L) V_{\mathcal{R}}$$

look at
the general
formula
→

← back to the first
Monday page

(33-34), Tuesday [2014-04-01] A1.3

9:00 - 11:00

 n_1 orthogonal to $\text{span}\{u_2, u_3\}$ n_2 orthogonal to $\text{span}\{u_3, u_1\}$ n_3 orthogonal to $\text{span}\{u_1, u_2\}$ \mathcal{F}_1 for face n_1, u_2, u_3 \mathcal{F} (not \mathcal{F})

$$(*) \quad M = A_{\mathcal{F}_1} t_1 \otimes u_1 + A_{\mathcal{F}_2} t_2 \otimes u_2 + A_{\mathcal{F}_3} t_3 \otimes u_3$$

$$M n_1 = A_{\mathcal{F}_1} t_1 (u_1 \cdot n_1) + A_{\mathcal{F}_2} t_2 (u_2 \cdot n_1) + A_{\mathcal{F}_3} t_3 (u_3 \cdot n_1)$$

\downarrow \downarrow \downarrow
 h_1 0 0

$$M n_1 = (A_{\mathcal{F}_1} h_1) t_1 = V_{\mathcal{F}_1} t_1$$

$$(*) \quad \int_{\mathcal{F}_1} t \cdot r \, dA = \int_{\mathcal{F}_1} t_1 \cdot r_0 \, dA + \int_{\mathcal{F}_1} t_1 \cdot L(x - p_0) \, dA$$

$$= A_{\mathcal{F}_1} t_1 \cdot r_0 + \int_{\mathcal{F}_1} t_1 \otimes (x - p_0) \cdot L \, dA = A_{\mathcal{F}_1} t_1 \cdot r_0 + A_{\mathcal{F}_1} t_1 \otimes (q_{\mathcal{F}_1} - p_0) \cdot L$$

[...]

 $\frac{1}{2} u_1$

Change of observer (frame-invariance)
Galilean transformations (Galilean group)

$$p_A^* = q_0^* + Q(p_A - p_0)$$

This transformation is defined by q_0^* and Q
 $q \equiv q_0^*$

$$\begin{aligned} \dot{p}_A^* &= \dot{q}_0^* + \dot{Q}(p_A - p_0) + Q(\dot{p}_A - \dot{p}_0) \\ &= \dot{q}_0^* + (\dot{Q} + QL)(p_A - p_0) = \dot{q}_0^* + (\dot{Q}Q^T + QLQ^T)(p_A^* - q_0^*) \end{aligned}$$

$$r_A^* = r_0^* + L^*(p_A^* - q_0^*) \quad p_0^* = q_0^*$$

$$F^*(r^*) = -(z^* \cdot r_0^* + T^* \cdot L^*) V_x$$

$$F^*(r^*) = F(r)$$

$$z^* \cdot r_0^* + T^* \cdot L^* = z \cdot r_0 + T \cdot L$$

where $r_0^* = \dot{q}_0^*$ an arbitrary vector \dot{q}

and $L^* = \dot{Q}Q^T + QLQ^T$ with arbitrary Q, \dot{Q}

$$\Rightarrow z^* \cdot r_0^* = z \cdot r_0 \quad \forall r_0, \forall \dot{q} \quad \Rightarrow z = 0, z^* = 0$$

$$\Rightarrow T^* \cdot (\dot{Q}Q^T + QLQ^T) = T \cdot L \quad \forall L, \forall \dot{Q}$$

$$\dot{Q} = 0 \quad \Rightarrow T^* \cdot QLQ^T = T \cdot L$$

$$z^* \cdot (r_0 + w_0^*) = z \cdot r_0 \quad \Rightarrow (z^* - z) \cdot r_0 = z^* \cdot w_0^*$$

[Notebook page scanned on 2014-06-21]

$$Q^T T^* Q \cdot L = T \cdot L \quad \forall L$$

$$\Rightarrow Q^T T^* Q = T$$

$$T^* = Q T Q^T$$

$$\dot{Q} \neq 0 \quad T^* \cdot \dot{Q} Q^T = 0 \quad \forall \dot{Q} Q^T \quad \Rightarrow \text{skw } T^* = 0$$

$$0 = \text{skw } T^* = \frac{1}{2} (Q T Q^T - Q T^T Q^T) = \frac{1}{2} Q (T - T^T) Q^T$$

$$\Rightarrow T = T^T$$

(35-36)₆ Wednesday [2014-04-02] A1.3
11:00 - 13:00

Summarizing affine motions

(affine bodies: bodies undergoing affine motions)

$$r(x) = r_0 + L(x - p_0)$$

$$F^{\text{ext}}(v) = f \cdot r_0 + M \cdot L$$

$$F^{\text{int}}(v) = -(z \cdot r_0 + T \cdot L) V_R$$

Balance

$$F(v) = 0 \quad \forall v$$

Objectivity

$$z = 0, \quad T = T^T$$

⇒

$$f = 0$$

$$\text{skw } M = 0$$

$$\text{sym } M = T V_R$$

with

$$T_{m_1} = t_1 \quad [\dots]$$

Balance equations in general

$$\mathcal{F}^{\text{ext}}(\boldsymbol{r}) = \int_{\mathcal{R}} \boldsymbol{b}(\boldsymbol{x}) \cdot \boldsymbol{r}(\boldsymbol{x}) \, dV + \int_{\partial\mathcal{R}} \boldsymbol{t}(\boldsymbol{x}) \cdot \boldsymbol{r}(\boldsymbol{x}) \, dA$$

$$\mathcal{F}^{\text{int}}(\boldsymbol{r}) = - \int_{\mathcal{R}} \boldsymbol{T}(\boldsymbol{x}) \cdot \underbrace{\boldsymbol{L}(\boldsymbol{x})}_{\nabla \boldsymbol{r}} \, dV$$

$$\mathcal{J}(\boldsymbol{r}) = \mathcal{F}^{\text{ext}}(\boldsymbol{r}) + \mathcal{F}^{\text{int}}(\boldsymbol{r}) = \int_{\mathcal{R}} \boldsymbol{b} \cdot \boldsymbol{r} \, dV + \int_{\partial\mathcal{R}} \boldsymbol{t} \cdot \boldsymbol{r} \, dA - \int_{\mathcal{R}} \boldsymbol{T} \cdot \nabla \boldsymbol{r} \, dV$$

$$\operatorname{div} \boldsymbol{r} = \operatorname{tr} \nabla \boldsymbol{r} \quad \nabla \boldsymbol{r} \text{ already defined}$$

$$\operatorname{div} \boldsymbol{T} \cdot \boldsymbol{a} = \operatorname{div} (\boldsymbol{T}^T \boldsymbol{a}) = \operatorname{tr} \nabla (\boldsymbol{T}^T \boldsymbol{a})$$

$$\operatorname{div} (\boldsymbol{T}^T \boldsymbol{r}) = \operatorname{tr} \nabla (\boldsymbol{T}^T \boldsymbol{r}) = \operatorname{div} \boldsymbol{T} \cdot \boldsymbol{r} + \boldsymbol{T} \cdot \nabla \boldsymbol{r}$$

$$\nabla (\boldsymbol{T}^T \boldsymbol{r}) \boldsymbol{e} = \lim_{h \rightarrow 0} \frac{\boldsymbol{T}^T(\boldsymbol{x} + h\boldsymbol{e}) \boldsymbol{r}(\boldsymbol{x} + h\boldsymbol{e}) - \boldsymbol{T}^T(\boldsymbol{x}) \boldsymbol{r}(\boldsymbol{x})}{h}$$

$$= \lim_{h \rightarrow 0} \frac{\boldsymbol{T}^T(\boldsymbol{x} + h\boldsymbol{e}) \boldsymbol{r}(\boldsymbol{x}) - \boldsymbol{T}^T(\boldsymbol{x}) \boldsymbol{r}(\boldsymbol{x})}{h} + \boldsymbol{T}^T(\boldsymbol{x}) \nabla \boldsymbol{r}(\boldsymbol{x}) \boldsymbol{e}$$

$$(21-22)_4 \Rightarrow \boldsymbol{r}(c(h)) = \boldsymbol{r}(c(0)) + \nabla \boldsymbol{r}(c(0)) (c(h) - c(0)) + o(h)$$

$$c(h) = \boldsymbol{x} + h\boldsymbol{e} \Rightarrow \boldsymbol{r}(\boldsymbol{x} + h\boldsymbol{e}) = \boldsymbol{r}(\boldsymbol{x}) + \nabla \boldsymbol{r}(\boldsymbol{x}) (h\boldsymbol{e}) + o(h)$$

$$w(x+he) := T^T(x+he) v(x)$$

$$\begin{aligned} \nabla(T^T v) e &= \lim_{h \rightarrow 0} \frac{w(x+he) - w(x)}{h} + T^T(x) \nabla w(x) e \\ &= \nabla w(x) e + T^T(x) \nabla v(x) e \end{aligned}$$

$$\text{tr} \nabla(T^T v) = \text{tr} \nabla w + \text{tr}(T^T \nabla v)$$

$$\text{tr} \nabla w = \text{div} T \cdot v \quad \begin{array}{l} \text{since } \text{tr} \nabla w \text{ is linear in } v(x) \\ \text{by the representation theorem} \end{array}$$

$$\text{div}(T^T v) = \text{tr} \nabla(T^T v) = \text{div} T \cdot v + \text{tr}(T^T \nabla v)$$

$$\text{div} T \cdot v = \text{div}(T^T v) - \text{tr}(T^T \nabla v)$$

$$-\int_{\mathcal{R}} T \cdot \nabla v \, dV = \int_{\mathcal{R}} \text{div} T \cdot v \, dV - \int_{\mathcal{R}} \text{div}(T^T v) \, dV$$

$$\int_{\mathcal{R}} \text{div} T \cdot v \, dV - \int_{\partial \mathcal{R}} T^T v \cdot n \, dA$$

$$= \int_{\mathcal{R}} \text{div} T \cdot v \, dV - \int_{\partial \mathcal{R}} T_m \cdot v \, dA$$

$$\mathcal{J}(v) = \int_{\mathcal{R}} b \cdot v \, dV + \int_{\partial \mathcal{R}} t \cdot v \, dA + \int_{\mathcal{R}} \text{div} T \cdot v \, dV - \int_{\partial \mathcal{R}} T_m \cdot v \, dA = 0$$

$$\text{CAUCHY BALANCE EQUATIONS} \quad \left\{ \begin{array}{l} b + \text{div} T = 0 \quad \mathcal{R} \\ t - T_m = 0 \quad \partial \mathcal{R} \end{array} \right.$$

(37-38)₇ Monday [2014-04-07]
16:00 - 18:00 413

Material response

$$T = \hat{T}(F) \quad \text{elastic material}$$

↑ response function

$$P_A = P_0 + F(\bar{P}_A - \bar{P}_0)$$

$$P_A^* = q_0^* + Q(P_A - P_0) \quad \text{change of observer}$$

$$P_0^* = q_0^* + Q(P_0 - P_0) = q_0^*$$

$$P_A^* - P_0^* = Q(P_A - P_0) = QF(\bar{P}_A - \bar{P}_0) = F^*(\bar{P}_A - \bar{P}_0)$$

$$T^* = QTQ^T$$

$$T^* = \hat{T}(F^*)$$

$$F^* = QF$$

$$\hat{T}(F^*) = Q\hat{T}(F)Q^T \quad \forall Q$$

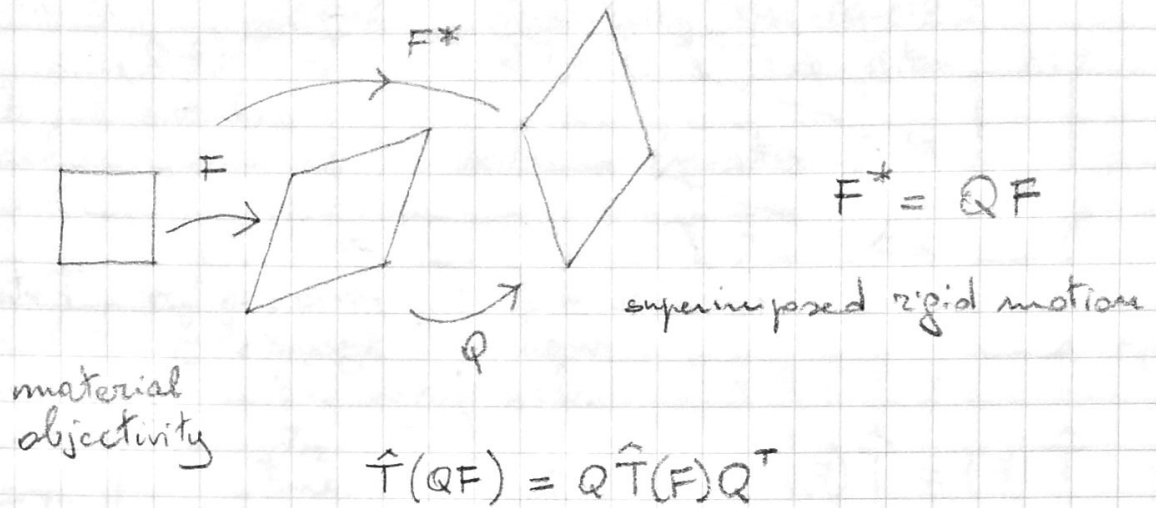
$$\hat{T}(QF) = Q\hat{T}(F)Q^T \quad \forall Q, \forall F$$

$$\hat{T}(QRU) = Q\hat{T}(RU)Q^T \quad \forall Q$$

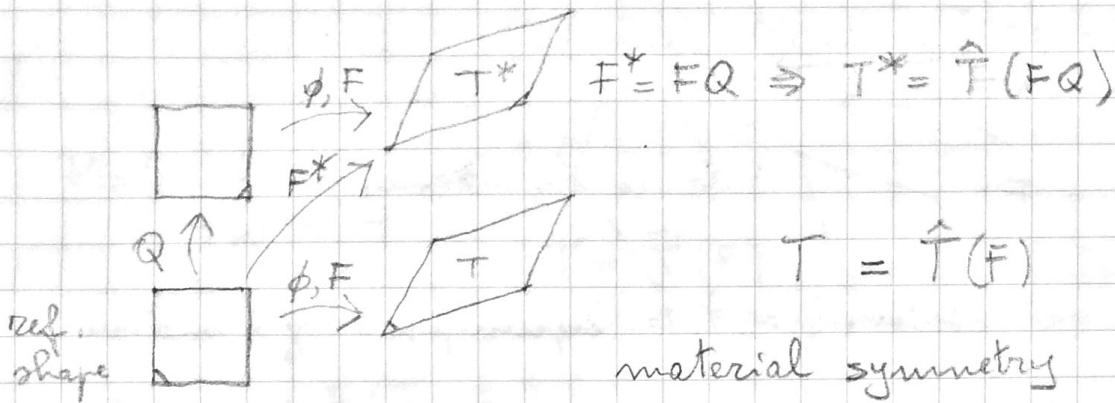
$$Q = R^T \Rightarrow \hat{T}(U) = R^T\hat{T}(F)R$$

$$\hat{T}(F) = R\hat{T}(U)R^T$$

objective response function reduced form



[Notebook page scanned on 2014-06-21]



$$\hat{T}(F) = \hat{T}(FQ) \quad \hat{T} \text{ is } Q \text{ invariant}$$

Symmetry group: collection of such rotations Q

Isotropic material: symmetry group $O(3)^+$

Isotropy

$$\hat{T}(F) \stackrel{\text{objectivity}}{=} \hat{T}(FQ) = \hat{T}(RUQ) = \hat{T}(\underbrace{RQ}_{\tilde{R}} \underbrace{Q^T U Q}_{\tilde{U}}) \quad \forall Q$$

$$= (RQ) \hat{T}(Q^T U Q) (RQ)^T = R \hat{T}(U) R^T$$

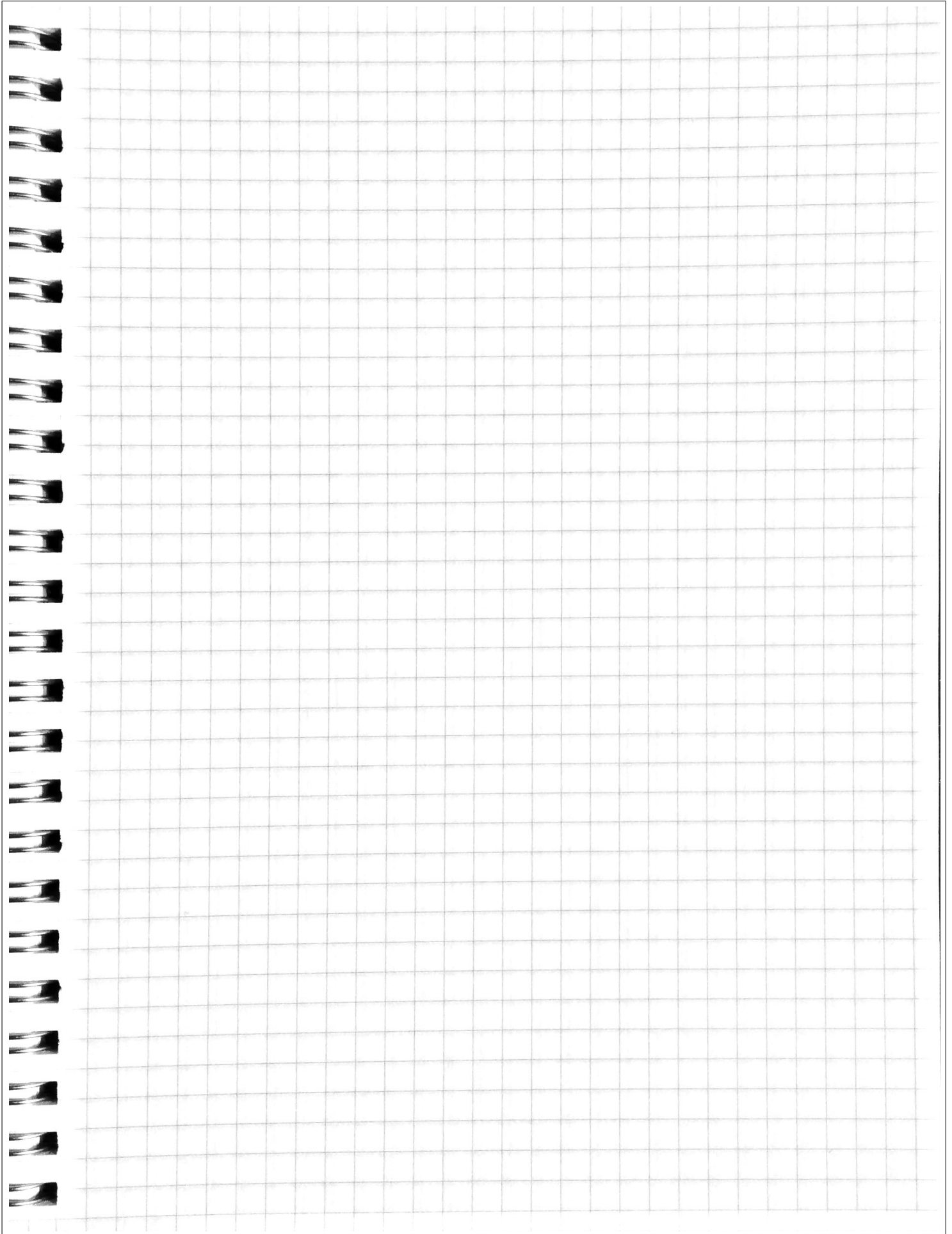
$$= R Q \hat{T}(Q^T U Q) Q^T R^T$$

$$\hat{T}(F) = \hat{T}(RU) = R \hat{T}(U) R^T$$

$$\Rightarrow Q \hat{T}(Q^T U Q) Q^T = \hat{T}(U)$$

$$\Leftrightarrow \hat{T}(Q^T U Q) = Q^T \hat{T}(U) Q \quad \forall Q$$

Isotropic response function



[Notebook page scanned on 2014-06-21]

(39-40)₇ Tuesday [2014-04-08]
9:00-11:00

$$T \cdot L V_R = T \cdot L V_R \det F$$

$$L (P_A - P_0) = LF (\bar{P}_A - \bar{P})$$

$$\begin{aligned} T \cdot L V_R \det F &= S \cdot LF V_R \\ &= SF^T \cdot L V_R \end{aligned}$$

$$T \det F = SF^T$$

$$S = TF^{-T} \det F$$

PIOLA stress

$$\int_R T \cdot \nabla_{\mathbf{r}} dV = \int_{\bar{R}} (T \cdot \nabla_{\mathbf{r}}) \det F dV = \int_{\bar{R}} S \cdot \nabla_{\bar{\mathbf{r}}} dV$$

$$\begin{aligned} \nabla_{\bar{\mathbf{r}}} &= \nabla_{\mathbf{r}} F \quad \Rightarrow \quad T \cdot \nabla_{\mathbf{r}} \det F = T \cdot \nabla_{\bar{\mathbf{r}}} F^{-1} \det F \\ &= (\det F) (TF^{-T}) \cdot \nabla_{\bar{\mathbf{r}}} = S \cdot \nabla_{\bar{\mathbf{r}}} \end{aligned}$$

$$\operatorname{div}(S^T \bar{r}) = S \cdot \nabla \bar{r} + \operatorname{div} S \cdot \bar{r}$$

fields on $\bar{\mathcal{R}}$

$$\int_{\mathcal{R}} b \cdot r \, dV = \int_{\bar{\mathcal{R}}} (b \cdot r) \det F \, dV = \int_{\bar{\mathcal{R}}} \bar{b} \cdot \bar{r} \, dV$$

$$\bar{r}(x) = r(\phi(x)), \quad \bar{b} = b \det F$$

$$\int_{\partial \mathcal{R}} t \cdot r \, dA = \int_{\partial \bar{\mathcal{R}}} \bar{t} \cdot \bar{r} \, d\Delta \quad \bar{t} = t \frac{\Delta_{\mathcal{R}}}{\Delta_{\bar{\mathcal{R}}}}$$

$$\int_{\mathcal{R}} T \cdot \nabla r \, dV = \int_{\bar{\mathcal{R}}} S \cdot \nabla \bar{r} \, d\Delta$$

$$\int_{\bar{\mathcal{R}}} \bar{b} \cdot \bar{r} \, dV + \int_{\partial \bar{\mathcal{R}}} \bar{t} \cdot \bar{r} \, d\Delta - \int_{\bar{\mathcal{R}}} S \cdot \nabla \bar{r} \, d\Delta = 0 \quad \forall \bar{r}$$

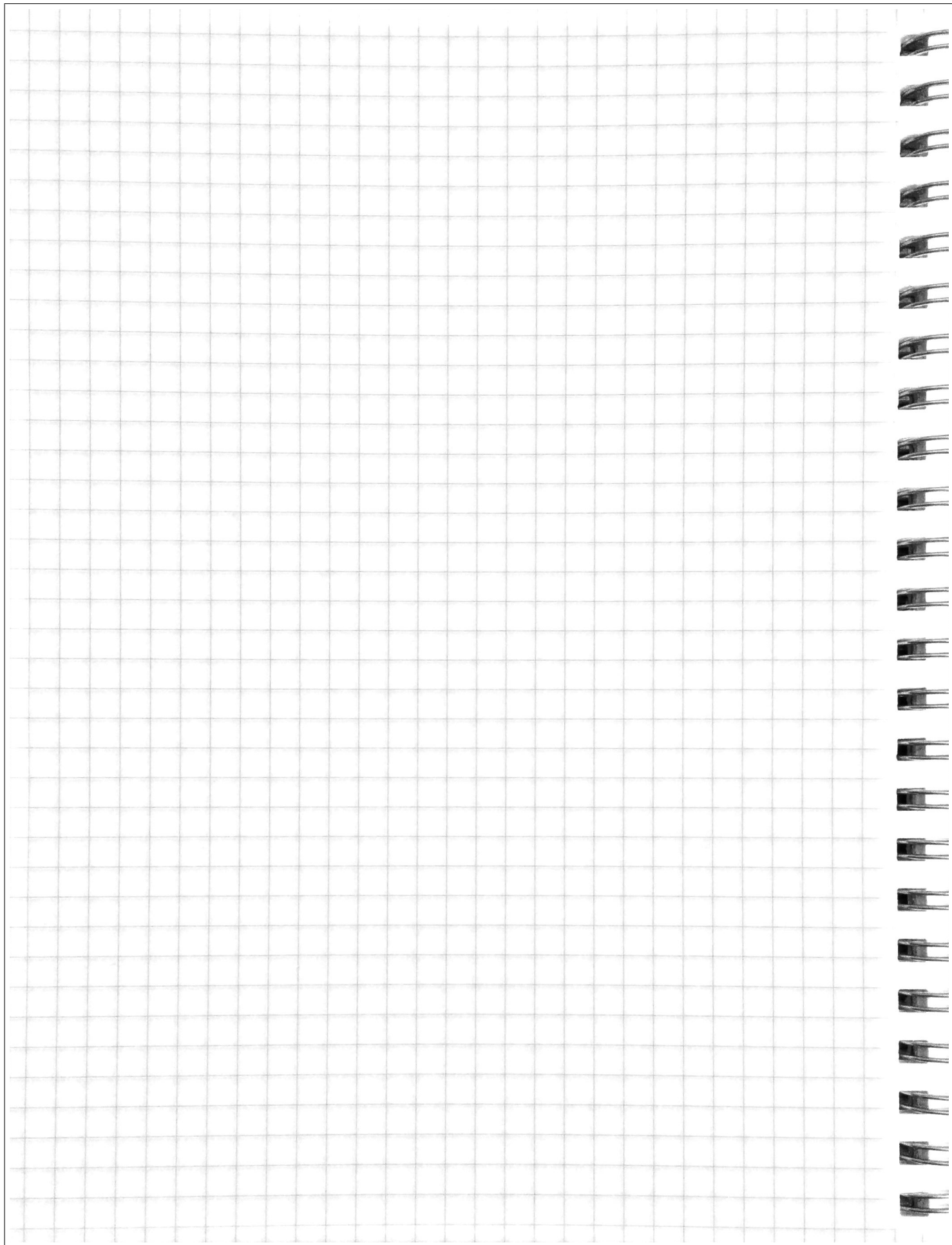
$$[\dots] \int_{\bar{\mathcal{R}}} \operatorname{div}(S^T \bar{r}) \, dV = \int_{\partial \bar{\mathcal{R}}} S^T \bar{r} \cdot \bar{n} \, d\Delta$$

↑ outward unit normal

$$[\dots] \begin{cases} \bar{b} + \operatorname{div} S = 0 & \bar{\mathcal{R}} \\ \bar{t} - S \bar{n} = 0 & \partial \bar{\mathcal{R}} \end{cases}$$

compare

$$t = T n, \quad \bar{t} = S \bar{n} \quad S = T \operatorname{cof} F$$



[Notebook page scanned on 2014-06-22]

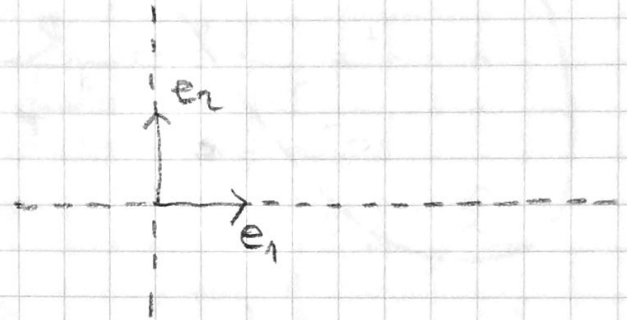
(41-42)₇ Wednesday [2014-04-09]

9:00 - 11:00 A1,2

Scalar form of the balance equations

$$\operatorname{div} T \cdot e_i = \operatorname{div}(T^T e_i) = \operatorname{tr} \nabla(T^T e_i)$$

$$[T] = \begin{pmatrix} \sigma_{11} & \sigma_{12} & \sigma_{13} \\ \sigma_{21} & \sigma_{22} & \sigma_{23} \\ \sigma_{31} & \sigma_{32} & \sigma_{33} \end{pmatrix}$$



$$T^T e_1 = \sigma_{11} e_1 + \sigma_{12} e_2 + \sigma_{13} e_3$$

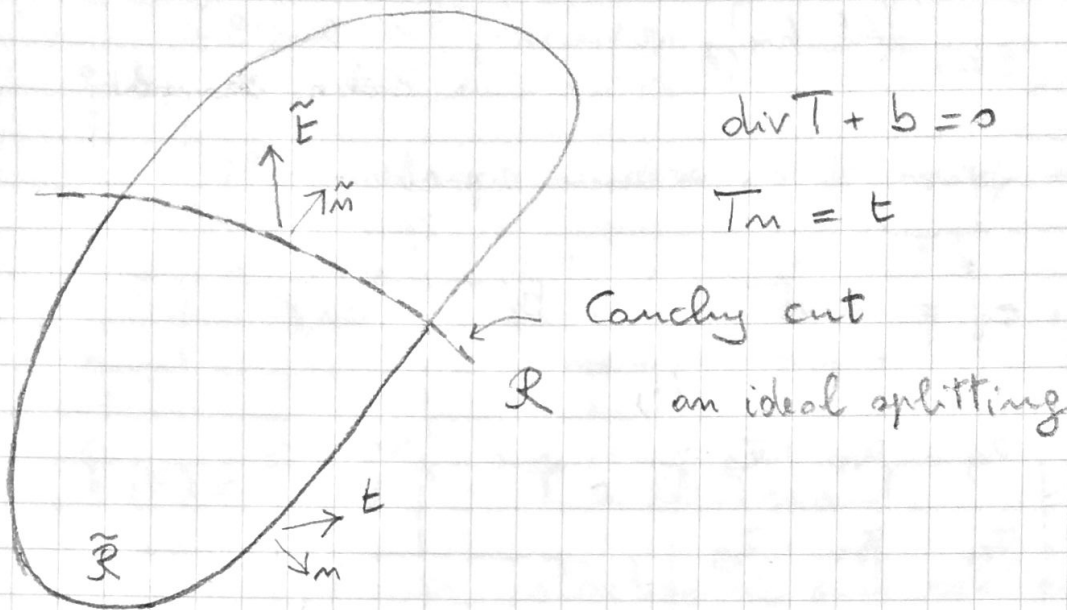
$$T^T e_2 = \sigma_{21} e_1 + \sigma_{22} e_2 + \sigma_{23} e_3$$

$$T^T e_3 = \sigma_{31} e_1 + \sigma_{32} e_2 + \sigma_{33} e_3$$

$$[\nabla(T^T e_1)] = \begin{pmatrix} \sigma_{11,1} & \sigma_{11,2} & \sigma_{11,3} \\ \sigma_{12,1} & \sigma_{12,2} & \sigma_{12,3} \\ \sigma_{13,1} & \sigma_{13,2} & \sigma_{13,3} \end{pmatrix}$$

$$\operatorname{div} T \cdot e_1 = \sigma_{11,1} + \sigma_{12,2} + \sigma_{13,3}$$

$$\operatorname{div} T \cdot e_i = \sigma_{i1,1} + \sigma_{i2,2} + \sigma_{i3,3}$$

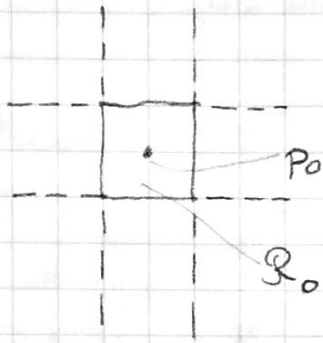


$$\int_{\tilde{\mathcal{R}}} b \cdot n \, dV + \int_{\partial \mathcal{R}} t \cdot n \, dA - \int_{\tilde{\mathcal{R}}} T \cdot \nabla n \, dV = 0$$

$$[III] \quad \text{div } T + b = 0$$

$$T_m = t, \quad T_{\tilde{m}} = \tilde{t}$$

\tilde{t} is the traction we should apply to keep the lower part of the body balanced after removing the upper part.



we can cut a sample
in the shape of a cube centered
at p_0 : $\mathcal{R}_0 \subset \mathcal{R}$

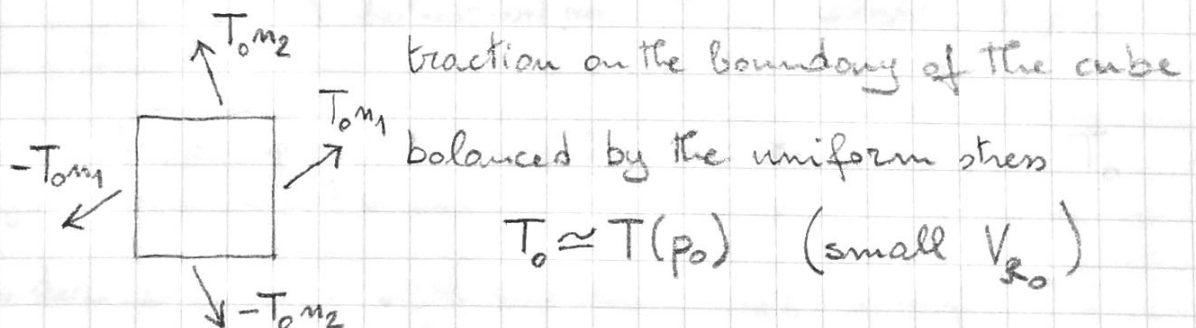
By using an affine test velocity field we get

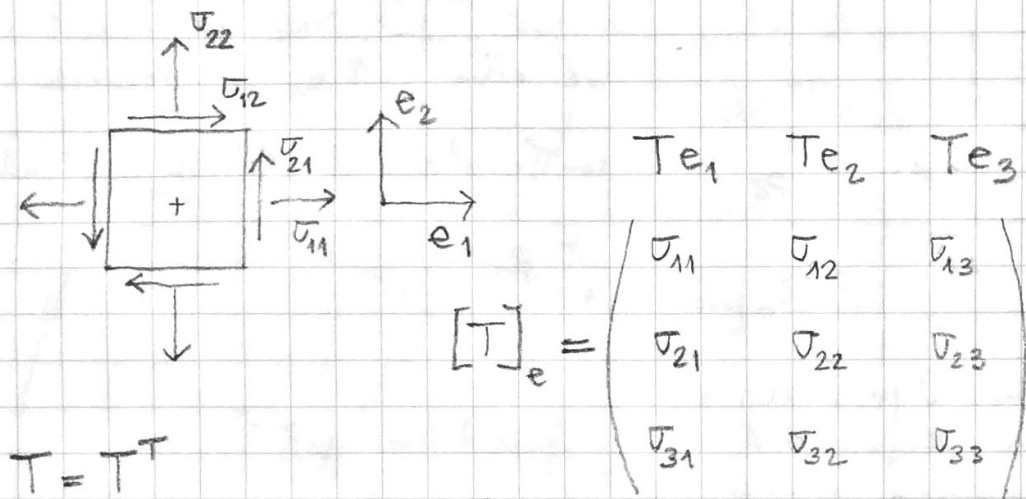
$$\int_{\mathcal{R}_0} \mathbf{T} \cdot \nabla \mathbf{r} \, dV = \int_{\mathcal{R}_0} \mathbf{T} \cdot \mathbf{L} \, dV = \left(\int_{\mathcal{R}_0} \mathbf{T} \, dV \right) \cdot \mathbf{L}$$

If we compare this expression with the one we
wrote for an affine body we find

stress in the
affine body $\mathbf{T}_0 \cdot \mathbf{L} \, V_{\mathcal{R}_0} = \int_{\mathcal{R}_0} \mathbf{T} \, dV \cdot \mathbf{L}$
stress field

$$\Rightarrow \mathbf{T}_0 = \frac{1}{V_{\mathcal{R}_0}} \int_{\mathcal{R}_0} \mathbf{T} \, dV \quad \text{mean stress}$$





$T = T^T$

$\Rightarrow \sigma_{21} = \sigma_{12} [\dots]$

↓
real eigenvalues and orthogonal eigenvectors

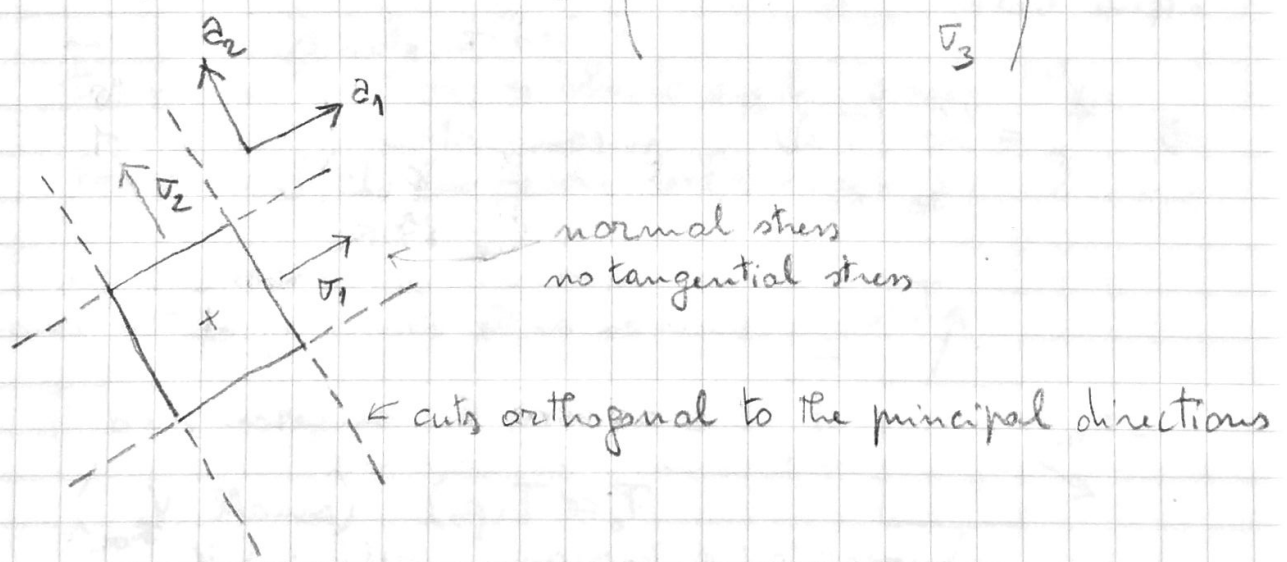
$\sigma_1, \sigma_2, \sigma_3$

principal stresses

a_1, a_2, a_3

principal directions

diagonal matrix $[T]_a = \begin{pmatrix} \sigma_1 & & \\ & \sigma_2 & \\ & & \sigma_3 \end{pmatrix}$



(43-44)₃ Monday [2014-04-14] A1.3
15:00-18:00

Elastic energy

stress power density per unit current volume $T \cdot \nabla v = T \cdot \dot{F} F^{-1}$

In an affine motion

$$\begin{aligned} T \cdot \dot{F} F^{-1} V_{\mathcal{R}} &= T \cdot \dot{F} F^{-1} V_{\mathcal{R}} \det F \\ &= T F^{-T} \cdot \dot{F} V_{\mathcal{R}} \det F \\ &= \underbrace{S \cdot \dot{F}}_{\text{power density per unit reference volume}} V_{\mathcal{R}} \end{aligned}$$

Hyperelastic material: there exists φ such that

$$\hat{T}(F) \cdot \dot{F} F^{-1} V_{\mathcal{R}} = \frac{d}{dt} \varphi(F) V_{\mathcal{R}}$$

in any (affine) motion

equivalently $\hat{S}(F) \cdot \dot{F} = \frac{d}{dt} \varphi(F)$

↑ elastic energy per unit reference volume

Objectivity: elastic energy invariance

under superposed rigid motion: $\varphi(F) = \varphi(QF)$

$$Q = R^T \Rightarrow \varphi(F) = \varphi(R^T R U) = \varphi(U)$$

necessary condition

Material symmetry

$$\varphi(F) = \varphi(FQ) \Rightarrow Q \text{ belongs to the symmetry group}$$

Isotropic material

$$\forall Q \quad \varphi(F) = \varphi(FQ) = \varphi(RUQ) = \varphi(\underbrace{RQ}_R \underbrace{QU}_U)$$

objectivity \downarrow

$$\varphi(U) = \varphi(Q^T U Q)$$

isotropic energy function

Spectral decomposition of the stretch

$$U = \lambda_1 P_1 + \lambda_2 P_2 + \lambda_3 P_3$$

unit eigenvector \downarrow
 $P_i = u_i \otimes u_i$

$$\forall Q \quad Q^T U Q = \lambda_1 Q^T P_1 Q + \lambda_2 Q^T P_2 Q + \lambda_3 Q^T P_3 Q$$

if $u = P_i u$ (eigenvector of U)

then $U u = \lambda_i u$

and $(Q^T U Q) \underbrace{Q^T u}_u = \lambda_i Q^T u$

\nwarrow eigenvector of $Q^T U Q$

isotropy $\varphi(U) = \varphi(Q^T U Q)$

same eigenvalues
different eigenvectors

$$\Rightarrow \varphi(U) \text{ is independent of the eigenvectors of } U$$

(45-46)₈ Tuesday [2014-04-15] A1.3
9:00-11:00

Isotropic elastic energy

$$\varphi(F) = \varphi(U) = \tilde{\varphi}(\lambda_1, \lambda_2, \lambda_3)$$

$$\begin{aligned} \varphi(U) &= \tilde{\varphi}(U^2) = \tilde{\varphi}(C) = \tilde{\varphi}(\lambda_1^2, \lambda_2^2, \lambda_3^2) \\ &= \hat{\varphi}(I_1, I_2, I_3) \end{aligned}$$

principal invariants of C \nearrow [iota]_3

coefficients of the characteristic polynomial

$$\det(C - \eta I) = \eta^3 - I_1 \eta^2 + I_2 \eta - I_3$$

$$I_1 = \text{tr } C$$

$$I_2 = \frac{1}{2} \left((\text{tr } C)^2 - \text{tr } C^2 \right)$$

$$I_3 = \det C$$

$$\begin{aligned} \frac{d}{dt} \varphi(F) &= \frac{d}{dt} \hat{\varphi}(I_1, I_2, I_3) \\ &= \hat{\varphi}_{,1} \frac{dI_1}{dt} + \hat{\varphi}_{,2} \frac{dI_2}{dt} + \hat{\varphi}_{,3} \frac{dI_3}{dt} \end{aligned}$$

$$\frac{d}{dt} L_1 = \frac{d}{dt} (F \cdot F) = 2F \cdot \dot{F} \quad L_1 = \text{tr}(F^T F) = F \cdot F$$

$$\frac{d}{dt} L_2 = 2L_1 F \cdot \dot{F} - \frac{1}{2} \frac{d}{dt} (C \cdot C)$$

$$= 2L_1 F \cdot \dot{F} - C \cdot \dot{C} = 2L_1 F \cdot \dot{F} - 2FC \cdot \dot{F}$$

$$C \cdot \dot{C} = C \cdot (\dot{F}^T F + F^T \dot{F}) = C \cdot ((F^T \dot{F})^T + F^T \dot{F})$$

$$= 2C \cdot \text{sym} F^T \dot{F} = 2C \cdot F^T \dot{F} = 2FC \cdot \dot{F}$$

$$\frac{d}{dt} L_3 = \frac{d}{dt} \det C = \frac{d}{dt} (\det F)^2 = 2 \det F \frac{d}{dt} \det F$$

$$= 2 \det F \det F \text{tr}(\dot{F} F^{-1}) = 2L_3 \text{tr}(\dot{F} F^{-1})$$

$$= 2L_3 \dot{F} \cdot F^{-T} = 2L_3 F^{-T} \cdot \dot{F}$$

$$= 2L_3 F C^{-1} \cdot \dot{F}$$

isotropic elastic energy

$$\frac{d}{dt} \varphi(F) = 2F \left(\left(\frac{\partial \hat{\varphi}}{\partial L_1} + \frac{\partial \hat{\varphi}}{\partial L_2} L_1 \right) I - \frac{\partial \hat{\varphi}}{\partial L_2} C + \frac{\partial \hat{\varphi}}{\partial L_3} L_3 C^{-1} \right) \cdot \dot{F}$$

$$\hat{\mathcal{S}}(F) \cdot \dot{F} = \frac{d}{dt} \varphi(F)$$

$$\hat{S}(F) \cdot \dot{F} = (\hat{T}(F) F^{-T} \det F) \cdot \dot{F} = \hat{T}(F) \cdot \dot{F} F^{-1} \det F$$

$$\hat{T}(F) = \hat{S}(F) F^T \frac{1}{\det F}$$

$$\hat{T}(F) = \frac{2}{\sqrt{l_3}} \left(\left(\frac{\partial \hat{\phi}}{\partial l_1} + \frac{\partial \hat{\phi}}{\partial l_2} l_1 \right) B - \frac{\partial \hat{\phi}}{\partial l_2} B^2 + \frac{\partial \hat{\phi}}{\partial l_3} l_3 I \right)$$

$B = FF^T$ left Cauchy-Green tensor

$$FC\dot{F}^T = FF^T FF^T = B^2$$

$$FC^{-1}F^T = F(F^{-1}F^{-T})F^T = I$$

Second Piola stress tensor

$$S \cdot \dot{F} = \frac{1}{2} S_{II} \cdot \dot{C}$$

$C = C^T \Rightarrow S_{II}$ is a symmetric tensor

$$\frac{1}{2} S_{II} \cdot \dot{C} = \frac{1}{2} S_{II} \cdot (\dot{F}^T F + F^T \dot{F}) = S_{II} \cdot \frac{1}{2} ((F^T \dot{F})^T + (F^T \dot{F}))$$

$$= S_{II} \cdot \text{sym} F^T \dot{F} = S_{II} \cdot F^T \dot{F} = F S_{II} \cdot \dot{F}$$

$$\Rightarrow S = F S_{II} \Rightarrow S_{II} = F^{-1} S = F^{-1} T F^{-T} \det F$$

(47-48)_g Wednesday [2014-04-16] A1.3
9:00-11:00

Force balance principle

$$\mathbb{F}^{\text{ext}}(v) + \mathbb{F}^{\text{int}}(v) = 0 \quad \forall v$$

Power expended

$$\int_{\mathcal{R}} b \cdot v \, dV + \int_{\partial \mathcal{R}} t \cdot v \, dA = \int_{\mathcal{R}} T \cdot \nabla v \, dV$$

by using the Piola stress

$$\int_{\mathcal{R}} T \cdot \nabla v \, dV = \int_{\mathcal{R}} T \cdot \dot{F} F^{-1} \, dV = \int_{\bar{\mathcal{R}}} S \cdot \dot{F} \, dV$$

Energy balance principle (dissipation inequality)

$$S \cdot \dot{F} - \frac{d}{dt} \varphi(F) \geq 0$$

$$S \cdot \dot{F} - \hat{S}(F) \cdot \dot{F} \geq 0$$

$$\underbrace{(S - \hat{S}(F))}_{S^+} \cdot \dot{F} \geq 0$$

$$S^+ \cdot \dot{F} \geq 0$$

$$\underbrace{(T - \hat{T}(F))}_{T^+} \cdot \dot{F} F^{-1} \geq 0$$

$$T^+ \cdot \dot{F} F^{-1} \geq 0$$

$$T = \hat{T}(F) + T^+$$

Possible choice (viscous stress)

$$T^+ := 2\mu \operatorname{sym} \dot{F} F^{-1}$$

$$(\operatorname{sym} \dot{F} F^{-1}) \cdot \dot{F} F^{-1} \geq 0$$

viscosity

$$\mu \geq 0 \Leftrightarrow T^+ \cdot \dot{F} F^{-1} \geq 0$$

Incompressible materials

$\det F = 1$ isochoric deformation (motion)

$$\frac{d}{dt} \det F = (\det F) \operatorname{tr}(\dot{F} F^{-1}) = 0$$

$$\Rightarrow \operatorname{tr}(\dot{F} F^{-1}) = 0 \Leftrightarrow \operatorname{tr} \nabla v = 0 \Leftrightarrow \operatorname{div} v = 0$$

$$a I \cdot \nabla v = a \operatorname{tr} \nabla v = 0 \quad \forall \text{ spherical tensor } a I$$

$$\text{sph } T := \frac{1}{3} (\operatorname{tr} T) I \Rightarrow (\text{sph } T) \cdot \nabla v = 0$$

$$\operatorname{dev} T := T - \text{sph } T \Rightarrow \operatorname{tr}(\operatorname{dev} T) = \operatorname{tr} T - \frac{1}{3} (\operatorname{tr} T) 3 = 0$$

$$T = \text{sph } T + \operatorname{dev} T$$

\uparrow spherical part \uparrow deviatoric part

$$\operatorname{tr} \nabla v = 0 \Rightarrow T \cdot \nabla v = (\text{sph } T + \operatorname{dev} T) \cdot \nabla v = \operatorname{dev} T \cdot \nabla v$$

\uparrow
 isochoric
 velocity field

$$\det F = 1 \Rightarrow \hat{S}(F) \cdot \dot{F} = \hat{T}(F) \cdot \dot{F} F^{-1} = \operatorname{dev} \hat{T}(F) \cdot \dot{F} F^{-1}$$

$$\text{elastic stress} \quad \operatorname{dev} \hat{T}(F) \cdot \dot{F} F^{-1} = \frac{d}{dt} \varphi(F)$$

$$\text{reactive stress} \quad \text{sph } T = -p I$$

\uparrow internal pressure

only the deviatoric part of the stress is determined
by the energy because of the incompressibility

$$\Rightarrow \underbrace{(\text{dev} T - \hat{T}(F)) \cdot \dot{F} F^{-1}} \geq 0$$

T^+ is a deviatoric tensor

$$T = \text{dev} T + \text{sph} T = \hat{T}(F) + T^+ - pI$$

$$T = \hat{T}(F) - pI + T^+$$

↑
elastic

↑
reactive

↑
dissipative

deviatoric

spherical

deviatoric

(49-50)₉ Wednesday [2014-04-23] A1.3

9:00-11:00

- A review about
- incompressibility
 - isochoric motions
 - deviatoric and spherical parts of the stress
 - dissipation and viscosity

A final remark about why T^+ should depend on ∇v in order to satisfy the assumption $T^+ \cdot \nabla v \geq 0$ in any motion.

(51-52)₁₀ Monday [2014-04-28] A1.3
16:00-18:00

Newtonian fluids

incompressibility $\operatorname{div} v(x, t) = 0$

$$\hat{T}(F) = 0$$

$$T^+ = 2\mu \operatorname{sym} \nabla v$$

$$\Rightarrow T = -pI + 2\mu \operatorname{sym} \nabla v$$

balance equation $\operatorname{div} T + b = 0$ $b = b_0 + b^{\text{in}}$

$$\operatorname{div} T = -\operatorname{div}(pI) + \mu(\operatorname{div} \nabla v + \operatorname{div} \nabla v^T)$$

$$\operatorname{div}(pI) = \nabla p \quad (\text{as a vector})$$

$$\operatorname{div} \nabla v = \Delta v \quad (\text{Laplacian})$$

$$\operatorname{div} \nabla v^T = \nabla(\operatorname{div} v) = 0$$

↓
0

$$\begin{aligned}\operatorname{div} p \mathbf{I} \cdot \mathbf{a} &= \operatorname{div} (p \mathbf{a}) = \operatorname{tr}(\nabla(p \mathbf{a})) \\ &= \operatorname{tr}(\mathbf{a} \otimes \nabla p) = \nabla p \cdot \mathbf{a}\end{aligned}$$

$$\begin{aligned}\nabla(p \mathbf{a}) \mathbf{e} &= \lim_{h \rightarrow 0} \frac{1}{h} (p(\mathbf{x} + h \mathbf{e}) - p(\mathbf{x})) \mathbf{a} \\ &= (\nabla p \cdot \mathbf{e}) \mathbf{a} = (\mathbf{a} \otimes \nabla p) \mathbf{e}\end{aligned}$$

$$\begin{aligned}\operatorname{tr}(\mathbf{u} \otimes \mathbf{v}) &= \frac{\operatorname{vol}(\mathbf{u} \otimes \mathbf{v}, \mathbf{e}_1, \mathbf{e}_2, \mathbf{e}_3)}{\operatorname{vol}(\mathbf{e}_1, \mathbf{e}_2, \mathbf{e}_3)} + \dots + \dots \\ &= \frac{\operatorname{vol}(\mathbf{u}(\mathbf{v} \cdot \mathbf{e}_1), \mathbf{e}_2, \mathbf{e}_3)}{\operatorname{vol}(\mathbf{e}_1, \mathbf{e}_2, \mathbf{e}_3)} + \dots + \dots \\ &= u_1(\mathbf{v} \cdot \mathbf{e}_1) + u_2(\mathbf{v} \cdot \mathbf{e}_2) + u_3(\mathbf{v} \cdot \mathbf{e}_3) \\ &= \mathbf{v} \cdot (u_1 \mathbf{e}_1 + u_2 \mathbf{e}_2 + u_3 \mathbf{e}_3) = \mathbf{v} \cdot \mathbf{u}\end{aligned}$$

$$\operatorname{div} \nabla \mathbf{r}^T \cdot \mathbf{a} = \operatorname{div}((\nabla \mathbf{r}) \mathbf{a}) = \operatorname{tr} \nabla \mathbf{w} \quad \mathbf{w} := (\nabla \mathbf{r}) \mathbf{a}$$

$$\begin{aligned}(\nabla \mathbf{w}) \mathbf{e} &= \lim_{h \rightarrow 0} \frac{1}{h} (\mathbf{w}(\mathbf{x} + h \mathbf{e}) - \mathbf{w}(\mathbf{x})) \\ &= \lim_{h \rightarrow 0} \frac{1}{h} (\nabla \mathbf{r}(\mathbf{x} + h \mathbf{e}) \mathbf{a} - \nabla \mathbf{r}(\mathbf{x}) \mathbf{a})\end{aligned}$$

$$\nabla v(x + h e) a = \lim_{d \rightarrow 0} \frac{1}{d} (v(x + h e + d a) - v(x + h e))$$

$$\nabla v(x) a = \lim_{d \rightarrow 0} \frac{1}{d} (v(x + d a) - v(x))$$

$$\begin{aligned} (\nabla w) e &= \lim_{d \rightarrow 0} \frac{1}{d} \left(\lim_{h \rightarrow 0} \frac{1}{h} (v(x + h e + d a) - v(x + d a)) \right. \\ &\quad \left. - \lim_{h \rightarrow 0} \frac{1}{h} (v(x + h e) - v(x)) \right) \end{aligned}$$

$$= \lim_{d \rightarrow 0} \frac{1}{d} (\nabla v(x + d a) e - \nabla v(x) e)$$

$$\Rightarrow \nabla w = \lim_{d \rightarrow 0} \frac{1}{d} (\nabla v(x + d a) - \nabla v(x))$$

$$\text{tr} \nabla w = \lim_{d \rightarrow 0} \frac{1}{d} (\text{tr}(\nabla v(x + d a)) - \text{tr}(\nabla v(x)))$$

$\text{div } v(x + d a) \quad \text{div } v(x)$

$$\text{tr} \nabla w = (\nabla(\text{div } v)) \cdot a$$

$$\Rightarrow \text{div} \nabla v^T = \nabla(\text{div } v)$$

$$\text{div } v = 0 \text{ (incompressibility)} \Rightarrow \text{div} \nabla v^T = 0$$

Finally let us define $\Delta v := \text{div} \nabla v$
 it is a vector field \uparrow Laplacian

$$\operatorname{div} \nabla v \cdot e_i = \operatorname{div}(\nabla v^T e_i) = \operatorname{tr} \nabla(\nabla v^T e_i)$$

$$\nabla v^T e_1 = v_{1,1} e_1 + v_{1,2} e_2 + v_{1,3} e_3$$

$$\nabla v^T e_i = v_{i,1} e_1 + v_{i,2} e_2 + v_{i,3} e_3$$

$$[\nabla(\nabla v^T e_i)] = \begin{pmatrix} v_{i,11} & v_{i,12} & v_{i,13} \\ v_{i,21} & v_{i,22} & v_{i,23} \\ v_{i,31} & v_{i,32} & v_{i,33} \end{pmatrix}$$

$$\operatorname{tr} \nabla(\nabla v^T e_i) = v_{i,11} + v_{i,22} + v_{i,33}$$

$$\operatorname{div} \nabla v = \Delta v \quad \text{Laplacian}$$

$$\operatorname{div} \nabla v^T \cdot e_i = \operatorname{div}(\nabla v e_i) = \operatorname{tr} \nabla(\nabla v e_i)$$

$$\nabla v e_1 = v_{1,1} e_1 + v_{2,1} e_2 + v_{3,1} e_3$$

$$\nabla v e_i = v_{1,i} e_1 + v_{2,i} e_2 + v_{3,i} e_3$$

$$[\nabla(\nabla v e_i)] = \begin{pmatrix} v_{1,i1} & v_{1,i2} & v_{1,i3} \\ v_{2,i1} & v_{2,i2} & v_{2,i3} \\ v_{3,i1} & v_{3,i2} & v_{3,i3} \end{pmatrix}$$

$$\begin{aligned} \operatorname{tr} \nabla(\nabla v e_i) &= v_{1,i1} + v_{2,i2} + v_{3,i3} \\ &= v_{1,1i} + v_{2,2i} + v_{3,3i} \\ &= \underbrace{(v_{1,1} + v_{2,2} + v_{3,3})}_{\operatorname{tr} \nabla v}, i \end{aligned}$$

$$\operatorname{tr} \nabla v = 0 \Rightarrow \operatorname{div} \nabla v^T = 0$$

(53-54)₁₀ Tuesday [2014-06-29] A1.3

9:00-11:00

Kinetic energy (density per unit current volume)

$$\frac{1}{2} \rho \|\mathbf{v}\|^2 = \frac{1}{2} \rho \mathbf{v} \cdot \mathbf{v}$$

$$K := \int_{\mathcal{R}} \frac{1}{2} \rho \|\mathbf{v}\|^2 dV = \int_{\bar{\mathcal{R}}} \frac{1}{2} (\det F) \rho \|\bar{\mathbf{v}}\|^2 dV \quad \text{total kinetic energy}$$

ρ_0 mass density per unit reference volume

$$= \int_{\bar{\mathcal{R}}} \frac{1}{2} \rho_0 \bar{\mathbf{v}} \cdot \bar{\mathbf{v}} dV$$

kinetic energy density per unit reference volume

Inertial force (density per unit current volume)

$$\int_{\mathcal{R}} \mathbf{b}^{\text{in}} \cdot \mathbf{v} dV = - \frac{d}{dt} \int_{\mathcal{R}} \frac{1}{2} \rho \|\mathbf{v}\|^2 dV$$

$$= - \frac{d}{dt} \int_{\bar{\mathcal{R}}} \frac{1}{2} \rho_0 \|\bar{\mathbf{v}}\|^2 dV = - \int_{\bar{\mathcal{R}}} \frac{d}{dt} \frac{1}{2} \rho_0 \bar{\mathbf{v}} \cdot \bar{\mathbf{v}} dV$$

independent of time \rightarrow constant in time

$$= - \int_{\bar{\mathcal{R}}} \rho_0 \bar{\mathbf{a}} \cdot \bar{\mathbf{v}} dV = - \int_{\mathcal{R}} \underbrace{\rho_0 \frac{1}{\det F}}_{\rho} \mathbf{a} \cdot \mathbf{v} dV$$

ρ mass density per unit current volume

$$\Rightarrow \mathbf{b}^{\text{in}}(\mathbf{x}) = -\rho(\mathbf{x}) \mathbf{a}(\mathbf{x})$$

$$\operatorname{div} T + b^{\text{in}} + b_0 = 0$$

$$-\nabla p + \mu \Delta v - \rho a + b_0 = 0$$

$$-\nabla p + \mu \Delta v - \rho ((\nabla v)v + v') + b_0 = 0$$

(velocity and acceleration spatial description)

↳ (21-22)

Navier-Stokes equation
for Newtonian fluids

(55-56)₁₀ Wednesday [2014-04-30] A1.3

9:00-11:00

In any motion, because of the balance principle,

$$\int_{\mathcal{R}} b \cdot r \, dV + \int_{\partial \mathcal{R}} t \cdot r \, dA = \int_{\mathcal{R}} T \cdot \nabla r \, dV$$

If $b = b^{\text{in}}$ and $t \cdot r = 0$ on $\partial \mathcal{R}$ then

$$\int_{\mathcal{R}} b^{\text{in}} \cdot r \, dV = \int_{\mathcal{R}} T \cdot \nabla r \, dV$$

with $\int_{\mathcal{R}} b^{\text{in}} \cdot r \, dV = - \frac{d}{dt} K$

and $\int_{\mathcal{R}} T \cdot \nabla r \, dV = \int_{\mathcal{R}} (\hat{T}(F) - pI + T^+) \cdot \nabla r \, dV$

Further if the material is incompressible then

$$pI \cdot \nabla r = p \operatorname{tr} \nabla r = p \operatorname{div} r = 0$$

otherwise we set $p=0$ because there is no reason

to split $\hat{T}(F)$ into a deviatoric and a spherical parts.

So whether the material is incompressible or not

$$\int_{\mathcal{R}} T \cdot \nabla r \, dV = \int_{\mathcal{R}} \hat{T}(F) \cdot \nabla r \, dV + \int_{\mathcal{R}} T^+ \cdot \nabla r \, dV$$

Recalling that for an hyperelastic material

$$\begin{aligned} \int_{\mathcal{R}} \hat{\mathbf{T}}(\mathbf{F}) \cdot \dot{\mathbf{F}} \mathbf{F}^{-1} dV &= \int_{\bar{\mathcal{R}}} \hat{\mathbf{S}}(\mathbf{F}) \cdot \dot{\mathbf{F}} dV \\ &= \int_{\bar{\mathcal{R}}} \frac{d}{dt} \varphi(\mathbf{F}) dV = \frac{d}{dt} \int_{\bar{\mathcal{R}}} \varphi(\mathbf{F}) dV \end{aligned}$$

denoting by

$$\Phi = \int_{\bar{\mathcal{R}}} \varphi(\mathbf{F}) dV$$

the total strain energy (or elastic energy)

we get

$$\int_{\mathcal{R}} \hat{\mathbf{T}}(\mathbf{F}) \cdot \nabla_{\mathbf{r}} dV = \frac{d}{dt} \Phi$$

Hence

$$\int_{\mathcal{R}} \mathbf{T} \cdot \nabla_{\mathbf{r}} dV = \frac{d}{dt} \Phi + \int_{\mathcal{R}} \mathbf{T}^+ \cdot \nabla_{\mathbf{r}} dV$$

By replacing these expressions in the starting power balance equation we get

$$-\frac{d}{dt} K - \frac{d}{dt} \Phi = \int_{\mathcal{R}} \mathbf{T}^+ \cdot \nabla_{\mathbf{r}} dV \geq 0$$

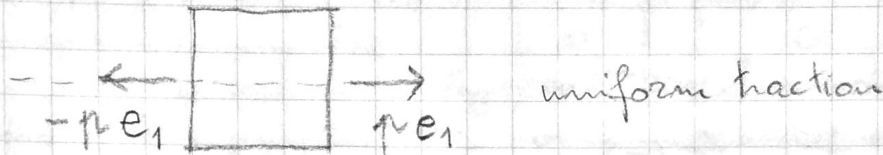
$$\Rightarrow \frac{d}{dt} K + \frac{d}{dt} \Phi \leq 0$$

↪ (47-48)₈

(57-58)₁₁ Monday [2014-05-05] A1.3

16:00 - 18:00

A standard problem in elasticity



cubic reference shape

Let us consider "cylindrical deformations" only:

affine deformations with

$$F e_1 = \lambda_1 e_1$$

$$F e_2 = \lambda_2 e_2$$

$$F e_3 = \lambda_3 e_3$$

$$[F] = \begin{pmatrix} \lambda_1 & & \\ & \lambda_2 & \\ & & \lambda_3 \end{pmatrix}$$

moment
tensor

$$M = \bar{l} F e_1 \otimes (p e_1) A_{F_1}$$

$$= \bar{l} \lambda_1 e_1 \otimes e_1 (p A_{F_1})$$

$$= l_1 e_1 \otimes e_1 (p l_2 l_3)$$

$$= \underbrace{l_1 l_2 l_3 p}_{\text{current volume } V_R} e_1 \otimes e_1$$

balance equations

$$f = 0$$

$$\text{skw } M = 0$$

$$\frac{M}{V_R} = T$$

from balance $T = p e_1 \otimes e_1$

material characterization

$$T = \hat{T}(F) - p I + T^d$$

\uparrow (incompressibility) \uparrow (dissipation)

Let us consider a compressible material with no dissipation:

$$T = \hat{T}(F)$$

with

$$[\hat{T}(F)] = \begin{pmatrix} \hat{\sigma}_{11}(F) & & \\ & \hat{\sigma}_{22}(F) & \\ & & \hat{\sigma}_{33}(F) \end{pmatrix}$$

$$[\dot{F} F^{-1}] = \begin{pmatrix} \frac{\dot{\lambda}_1}{\lambda_1} & & \\ & \frac{\dot{\lambda}_2}{\lambda_2} & \\ & & \frac{\dot{\lambda}_3}{\lambda_3} \end{pmatrix}$$

balance

$$\begin{pmatrix} \sigma_{11} & & \\ & \sigma_{22} & \\ & & \sigma_{33} \end{pmatrix} = \begin{pmatrix} p & & \\ & 0 & \\ & & 0 \end{pmatrix}$$

$$T = p e_1 \otimes e_1$$

A simple choice for $\hat{T}(F)$ is

$$\hat{\sigma}_{11}(F) = a_1(\lambda_1 - 1)$$

$$\hat{\sigma}_{22}(F) = a_2(\lambda_2 - 1)$$

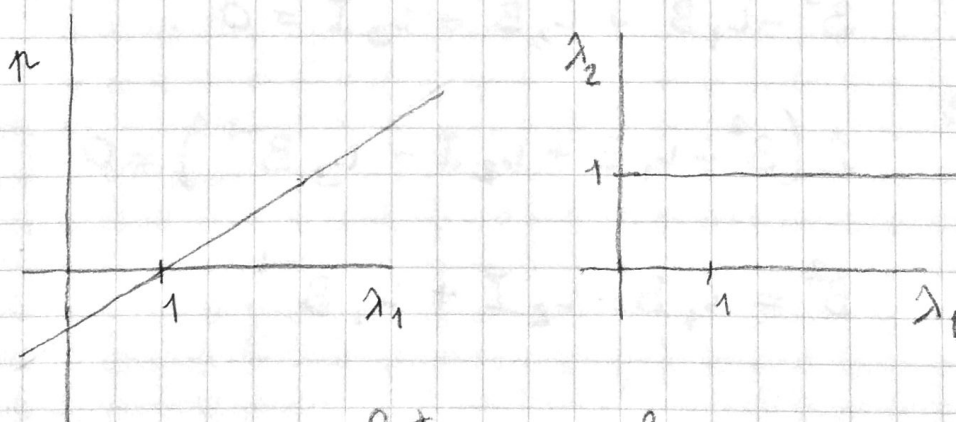
$$\hat{\sigma}_{33}(F) = a_3(\lambda_3 - 1)$$

Balance and material response

$$p = a_1(\lambda_1 - 1)$$

$$0 = a_2(\lambda_2 - 1)$$

$$0 = a_3(\lambda_3 - 1)$$



solution graphs

(59-60)₁₁ Tuesday [2014-05-06] A1.3
9:00-11:00

The solution we derived from the assumed material response is unsatisfactory because it allows λ_1 to be negative; further λ_2 and λ_3 are always equal to 1.

Let us consider the response function for a hyperelastic material $\rightarrow (45-46)_8$

$$\hat{T}(F) = \frac{2}{\sqrt{I_3}} \left(\left(\frac{\partial \hat{\psi}}{\partial I_1} + \frac{\partial \hat{\psi}}{\partial I_2} I_1 \right) B - \frac{\partial \hat{\psi}}{\partial I_2} B^2 + \frac{\partial \hat{\psi}}{\partial I_3} I_3 I \right)$$

with $B = FF^T$ (left Cauchy-Green tensor)

We can get a new expression by using the Cayley-Hamilton theorem stating that any tensor satisfies its characteristic equation:

$$B^3 - I_1 B^2 + I_2 B - I_3 I = 0$$

We get

$$B(B^2 - I_1 B + I_2 I - I_3 B^{-1}) = 0$$

$$B^2 = I_1 B - I_2 I + I_3 B^{-1}$$

$$\hat{T}(F) = \frac{2}{\sqrt{L_3}} \left(\frac{\partial \hat{\varphi}}{\partial L_1} B + \left(\frac{\partial \hat{\varphi}}{\partial L_3} L_3 + \frac{\partial \hat{\varphi}}{\partial L_2} L_2 \right) I - \frac{\partial \hat{\varphi}}{\partial L_2} L_3 B^{-1} \right)$$

If

$$[F] = \begin{pmatrix} \lambda_1 & & \\ & \lambda_2 & \\ & & \lambda_3 \end{pmatrix}$$

then


$$[B] = \begin{pmatrix} \lambda_1^2 & & \\ & \lambda_2^2 & \\ & & \lambda_3^2 \end{pmatrix}$$

(61-62)₁₁ Wednesday [2014-05-07] A1.3

9:00 - 11:00

let us consider an incompressible material.

The motion will be isochoric: any deformation is characterized by $\det F = 1$.



$$[F] = \begin{pmatrix} \lambda & & \\ & \frac{1}{\sqrt{\lambda}} & \\ & & \frac{1}{\sqrt{\lambda}} \end{pmatrix}$$

A standard strain energy for so called rubber-like material is

$$\varphi_H(F) = \kappa_1 (I_1 - 3) \quad \text{neo-Hookean}$$

or

$$\varphi_{MR}(F) = \kappa_1 (I_1 - 3) + \kappa_2 (I_2 - 3)$$

Mooney-Rivlin

For the assumed deformation gradient we have

$$[C] = [F^T F] = \begin{pmatrix} \lambda^2 & & \\ & \frac{1}{\lambda} & \\ & & \frac{1}{\lambda} \end{pmatrix}$$

$$I_1 = \text{tr } C = \lambda^2 + \frac{2}{\lambda}$$

$$[C^2] = \begin{pmatrix} \lambda^4 & & \\ & \frac{1}{\lambda^2} & \\ & & \frac{1}{\lambda^2} \end{pmatrix}$$

$$v_2 = \frac{1}{2} \left((\text{tr } C)^2 - \text{tr } C^2 \right) = \frac{1}{2} \left(\lambda^4 + 4\lambda + \frac{4}{\lambda^2} - \lambda^4 - \frac{2}{\lambda^2} \right) = 2\lambda + \frac{1}{\lambda^2}$$

$$v_3 = 1$$

$$[\dot{F}] = \begin{pmatrix} \dot{\lambda} & & \\ & -\frac{1}{2} \lambda^{-\frac{3}{2}} \dot{\lambda} & \\ & & -\frac{1}{2} \lambda^{-\frac{3}{2}} \dot{\lambda} \end{pmatrix}$$

$$[\dot{F} F^{-1}] = \begin{pmatrix} \frac{\dot{\lambda}}{\lambda} & & \\ & -\frac{1}{2} \lambda^{-\frac{3}{2}} \dot{\lambda} \lambda^{-\frac{1}{2}} & \\ & & -\frac{1}{2} \lambda^{-\frac{3}{2}} \dot{\lambda} \lambda^{-\frac{1}{2}} \end{pmatrix}$$

$$[\dot{F} F^{-1}] = \begin{pmatrix} 1 & & \\ & -\frac{1}{2} & \\ & & -\frac{1}{2} \end{pmatrix} \frac{\dot{\lambda}}{\lambda}$$

note $\text{tr } \dot{F} F^{-1} = 0$

(63-64)₁₂ Monday [2014-05-12] A13
16:00-18:00

$$\hat{T}(F) \cdot \dot{F} F^{-1} = \left(\hat{\sigma}_1 - \frac{1}{2} (\hat{\sigma}_2 + \hat{\sigma}_3) \right) \frac{\dot{\lambda}}{\lambda}$$

for a neo-Hookean material

$$\frac{d}{dt} \varphi(F) = c_1 \left(2 \lambda \dot{\lambda} - \frac{2}{\lambda^2} \dot{\lambda} \right) = 2 c_1 \left(\lambda^2 - \frac{1}{\lambda} \right) \frac{\dot{\lambda}}{\lambda}$$

Setting $\hat{\sigma}_0 := \hat{\sigma}_1 - \frac{1}{2} (\hat{\sigma}_2 + \hat{\sigma}_3)$

from $\hat{T}(F) \cdot \dot{F} F^{-1} = \frac{d}{dt} \varphi(F)$

we get $\hat{\sigma}_0 = 2 c_1 \left(\lambda^2 - \frac{1}{\lambda} \right)$

$$[\text{dev } \hat{T}(F)] = \begin{pmatrix} \hat{\sigma}_1^D \\ \hat{\sigma}_2^D \\ \hat{\sigma}_3^D \end{pmatrix}$$

$$\hat{\sigma}_1^D = \hat{\sigma}_1 - \frac{1}{3} (\hat{\sigma}_1 + \hat{\sigma}_2 + \hat{\sigma}_3) = \frac{2}{3} \left(\hat{\sigma}_1 - \frac{1}{2} (\hat{\sigma}_2 + \hat{\sigma}_3) \right)$$

$$\hat{\sigma}_1^D = \frac{2}{3} \hat{\sigma}_0$$

$$\begin{pmatrix} \sigma_1 \\ \sigma_2 \\ \sigma_3 \end{pmatrix} = \begin{pmatrix} \hat{\sigma}_1^D \\ \hat{\sigma}_2^D \\ \hat{\sigma}_3^D \end{pmatrix} - p \begin{pmatrix} 1 \\ 1 \\ 1 \end{pmatrix}$$

$$T = \text{dev } T + \text{sph } T$$

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$$\sigma_1 = \hat{\sigma}_1^D - p$$

$$\sigma_2 = \hat{\sigma}_2^D - p$$

$$\sigma_3 = \hat{\sigma}_3^D - p$$

$$\sigma_1 = \mu$$

$$\sigma_2 = 0$$

$$\sigma_3 = 0$$

$$T = \hat{T}(F) - pI$$

$$T = \frac{M}{V_R}$$

material
characterization

balance equations

$$\hat{\sigma}_1^D - p = \mu$$

$$\hat{\sigma}_2^D - p = 0$$

$$\hat{\sigma}_3^D - p = 0$$

$$0 - 3p = \mu$$

(taking the trace)

$$\Rightarrow p = -\frac{1}{3}\mu$$

$$\hat{\sigma}_1^D = \mu + p = \frac{2}{3}\mu$$

$$\hat{\sigma}_2^D = p = -\frac{1}{3}\mu$$

$$\hat{\sigma}_3^D = p = -\frac{1}{3}\mu$$

$$\hat{\sigma}_1^D = \frac{2}{3} \hat{\sigma}_0^D \Rightarrow$$

$$\frac{2}{3} \hat{\sigma}_0^D = \frac{2}{3} \mu$$

$$2c_1 \left(\lambda^2 - \frac{1}{\lambda} \right) = \mu$$

(65-66)₁₂ Tuesday [2014-05-13] 41.3
9:00-11:00

$$T = \hat{T}(F) - pI + T^+$$

$$T^+, \dot{F}F^{-1} \geq 0$$

dissipation principle

The standard choice for a viscous material is

$$T^+ = 2\mu \operatorname{sym} \dot{F}F^{-1}$$

with $\mu \geq 0$ by the dissipation principle.

$$[T^+] = \begin{pmatrix} \sigma_1^+ & & \\ & \sigma_2^+ & \\ & & \sigma_3^+ \end{pmatrix} = 2\mu \begin{pmatrix} 1 & & \\ & -\frac{1}{2} & \\ & & -\frac{1}{2} \end{pmatrix} \frac{\dot{\lambda}}{\lambda}$$

$$\sigma_1 = \hat{\sigma}_1^D - p + \sigma_1^+$$

$$\sigma_1 = \mu$$

$$\sigma_2 = \hat{\sigma}_2^D - p + \sigma_2^+$$

$$\sigma_2 = 0$$

$$\sigma_3 = \hat{\sigma}_3^D - p + \sigma_3^+$$

$$\sigma_3 = 0$$

material
characterization

balance
equations

$$\hat{\sigma}_1^D - p + \sigma_1^+ = \mu$$

$$\hat{\sigma}_2^D - p + \sigma_2^+ = 0$$

$$\hat{\sigma}_3^D - p + \sigma_3^+ = 0$$

$$0 - 3p + 0 = \mu \quad \Rightarrow \quad p = -\frac{1}{3}\mu$$

$$\hat{\sigma}_1^D - p + \sigma_1^+ = \mu$$

$$\frac{2}{3} \hat{\sigma}_0 + \sigma_1^+ = \mu + p = \frac{2}{3} \mu$$

$$\hat{\sigma}_0 + \frac{3}{2} \sigma_1^+ = \mu$$

$$2 \kappa_1 \left(\lambda^2 - \frac{1}{\lambda} \right) + 3 \mu \frac{\dot{\lambda}}{\lambda} = \mu$$

Find a solution for a given value μ_0 .

Let λ_0 be such that

$$2 \kappa_1 \left(\lambda_0^2 - \frac{1}{\lambda_0} \right) = \mu_0$$

and let us look for a solution close to λ_0

$$\begin{aligned} \lambda(t) &= \lambda_0 + \tilde{\varepsilon}(t) \quad \text{with } \tilde{\varepsilon} \text{ a small} \\ &= 1 + \underbrace{\varepsilon_0 + \tilde{\varepsilon}(t)}_{\varepsilon(t)} \quad \text{quantity } \forall t \\ &= 1 + \varepsilon(t) \end{aligned}$$

$$\Rightarrow \dot{\lambda}(t) = \dot{\tilde{\varepsilon}}(t) = \dot{\varepsilon}(t)$$

We assume the motion is so slow that even $\dot{\tilde{\varepsilon}}$ is a small quantity $\forall t$.

$$\lambda^2 = (\lambda_0 + \tilde{\varepsilon})^2 = \lambda_0^2 + 2\lambda_0 \tilde{\varepsilon} + o(\tilde{\varepsilon})$$

$$\frac{1}{\lambda} = \frac{1}{\lambda_0} - \frac{1}{\lambda_0^2} \tilde{\varepsilon} + o(\tilde{\varepsilon})$$

$$\frac{\dot{\lambda}}{\lambda} = \left(\frac{1}{\lambda_0} - \frac{1}{\lambda_0^2} \tilde{\varepsilon} \right) \dot{\tilde{\varepsilon}} + o(\tilde{\varepsilon}) = \frac{1}{\lambda_0} \dot{\tilde{\varepsilon}} + o(\tilde{\varepsilon})$$

$$2c_1 \left(\lambda_0^2 + 2\lambda_0 \tilde{\varepsilon} - \frac{1}{\lambda_0} + \frac{1}{\lambda_0^2} \tilde{\varepsilon} \right) + 3\mu \frac{1}{\lambda_0} \dot{\tilde{\varepsilon}} = p_0$$

$$\underbrace{2c_1 \left(\lambda_0^2 - \frac{1}{\lambda_0} \right)}_{p_0} + 2c_1 \left(2\lambda_0 + \frac{1}{\lambda_0^2} \right) \tilde{\varepsilon} + 3\mu \frac{1}{\lambda_0} \dot{\tilde{\varepsilon}} = p_0$$

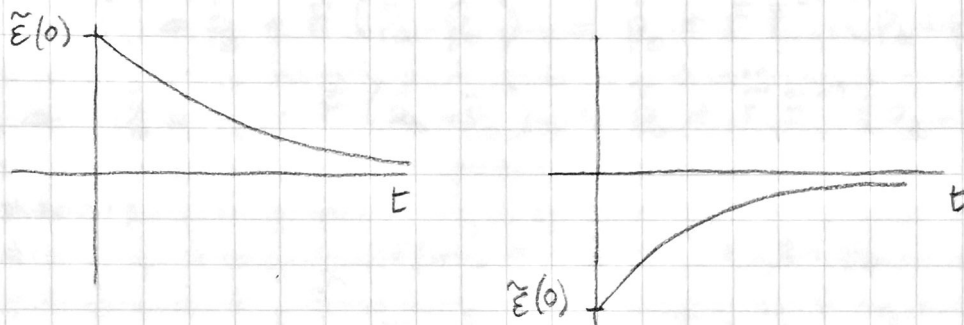
$$2c_1 \left(2\lambda_0 + \frac{1}{\lambda_0^2} \right) \tilde{\varepsilon} + 3\mu \dot{\tilde{\varepsilon}} = 0$$

(67-68)₁₂ Wednesday [2014-05-14]A1.3
9:00-11:00

$$a \tilde{\epsilon}(t) + \dot{\tilde{\epsilon}}(t) = 0$$

$$a := \frac{2-c_1}{3\mu} \left(2\lambda_0^2 + \frac{1}{\lambda_0} \right)$$

$$\tilde{\epsilon}(t) = \tilde{\epsilon}(0) e^{-at}$$

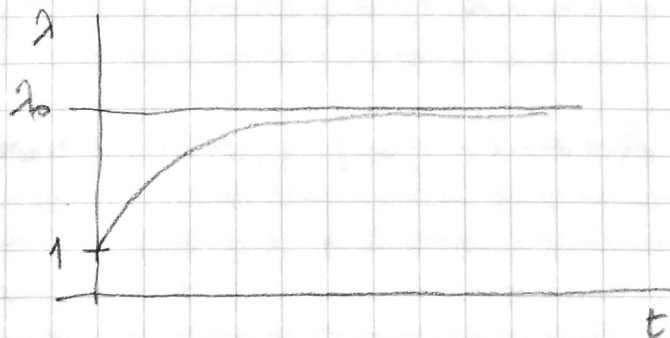


If ϵ_0 is very small we can set

$$\tilde{\epsilon}(0) = -\epsilon_0$$

$$\Rightarrow \lambda(0) = 1 + \epsilon_0 + \tilde{\epsilon}(0) = 1$$

$$\lambda(t) = 1 + \epsilon_0 (1 - e^{-at})$$



[Notebook page scanned on 2014-06-21]

The viscosity makes the deformation evolve in time. If the traction p is applied suddenly at time $t=0$ the body starts deforming and will approach the final shape with a decreasing velocity as $t \rightarrow \infty$.

[Notebook page scanned on 2014-06-21]

(69-70)₁₃ Monday [2014-05-19] A1.3
16:00-18:00

Inertial forces and free oscillations

$$b^{\text{in}}(x) = -\rho(x) a(x) \quad \rightarrow (53-54)_{10}$$

Affine deformations

$$p_A(t) = p_0(t) + F(t)(\bar{p}_A - \bar{p}_0)$$

$$\dot{r}_A = \dot{p}_A = \dot{p}_0 + \dot{F}(\bar{p}_A - \bar{p}_0) = \dot{p}_0 + \dot{F}F^{-1}(p_A - p_0)$$

$$a_A = \ddot{p}_A = \ddot{p}_0 + \ddot{F}(\bar{p}_A - \bar{p}_0) = \ddot{p}_0 + \ddot{F}F^{-1}(p_A - p_0)$$

test velocity
field

$$v(x) = v_0 + \nabla v(x - p_0) \quad x \in \mathcal{R}$$

$$\bar{v}(x) = v_0 + \nabla \bar{v}(x - \bar{p}_0) \quad x \in \bar{\mathcal{R}}$$

acceleration
field

$$a(x) = a_0 + \ddot{F}F^{-1}(x - p_0) \quad x \in \mathcal{R}$$

$$\bar{a}(x) = a_0 + \ddot{F}(x - \bar{p}_0) \quad x \in \bar{\mathcal{R}}$$

$$\mathcal{F}^{\text{in}}(v) = \int_{\mathcal{R}} b^{\text{in}}(x) \cdot v(x) dV = - \int_{\bar{\mathcal{R}}} \rho_0 \bar{a}(x) \cdot \bar{v}(x) dV$$

$$= - \int_{\bar{\mathcal{R}}} \rho_0 (a_0 + \ddot{F}(x - \bar{p}_0)) \cdot (v_0 + \nabla \bar{v}(x - \bar{p}_0)) dV$$

$$\begin{aligned} \mathcal{F}^{in}(\mathcal{r}) &= - \int_{\bar{\mathcal{R}}} \rho_0 \mathbf{a}_0 \cdot \boldsymbol{\nu}_0 dV - \int_{\bar{\mathcal{R}}} \rho_0 \mathbf{a}_0 \cdot \nabla \bar{\mathbf{r}} (\mathbf{x} - \bar{\mathbf{p}}_0) dV \\ &\quad - \int_{\bar{\mathcal{R}}} \rho_0 \ddot{\mathbf{F}} (\mathbf{x} - \bar{\mathbf{p}}_0) \cdot \boldsymbol{\nu}_0 dV - \int_{\bar{\mathcal{R}}} \rho_0 \ddot{\mathbf{F}} (\mathbf{x} - \bar{\mathbf{p}}_0) \cdot \nabla \bar{\mathbf{r}} (\mathbf{x} - \bar{\mathbf{p}}_0) dV \end{aligned}$$

The barycenter of the reference shape is the position \mathbf{x}_G such that

$$(\mathbf{x}_G - \bar{\mathbf{p}}_0) = \frac{1}{V_{\bar{\mathcal{R}}}} \int_{\bar{\mathcal{R}}} \rho_0 (\mathbf{x} - \bar{\mathbf{p}}_0) dV$$

We choose $\bar{\mathbf{p}}_0 \equiv \mathbf{x}_G$ so that $\int_{\bar{\mathcal{R}}} \rho_0 (\mathbf{x} - \bar{\mathbf{p}}_0) dV = 0$
and

$$\mathcal{F}^{in}(\mathcal{r}) = - \int_{\bar{\mathcal{R}}} \rho_0 dV \mathbf{a}_0 \cdot \boldsymbol{\nu}_0$$

$$- \int_{\bar{\mathcal{R}}} \rho_0 \ddot{\mathbf{F}} (\mathbf{x} - \bar{\mathbf{p}}_0) \otimes (\mathbf{x} - \bar{\mathbf{p}}_0) \cdot \nabla \bar{\mathbf{r}} dV$$

$$= -m \mathbf{a}_0 \cdot \boldsymbol{\nu}_0 - \ddot{\mathbf{F}} \underbrace{\int_{\bar{\mathcal{R}}} \rho_0 (\mathbf{x} - \bar{\mathbf{p}}_0) \otimes (\mathbf{x} - \bar{\mathbf{p}}_0) dV}_{\mathcal{J}} \cdot \nabla \bar{\mathbf{r}}$$

↑
 total mass

J
 Euler tensor

$$\begin{aligned}
 \mathfrak{J}^{\text{in}}(\mathbf{r}) &= -m \mathbf{a}_0 \cdot \mathbf{r}_0 - \ddot{\mathbf{F}} \bar{\mathbf{J}} \cdot \nabla \mathbf{r} \\
 &= -m \mathbf{a}_0 \cdot \mathbf{r}_0 - \ddot{\mathbf{F}} \bar{\mathbf{J}} \cdot (\nabla \mathbf{r} \mathbf{F}) \\
 &= -m \mathbf{a}_0 \cdot \mathbf{r}_0 - \ddot{\mathbf{F}} \bar{\mathbf{J}} \mathbf{F}^T \cdot \nabla \mathbf{r} = \mathbf{f}^{\text{in}} \cdot \mathbf{r}_0 + \mathbf{M}^{\text{in}} \cdot \nabla \mathbf{r}
 \end{aligned}$$

By using coordinates as in

$$\mathbf{x} - \bar{\mathbf{p}}_0 = s_1 \mathbf{e}_1 + s_2 \mathbf{e}_2 + s_3 \mathbf{e}_3$$

we can compute the matrix of $\bar{\mathbf{J}}$ for uniform ρ_0

$$[\bar{\mathbf{J}}]_{ij} = \rho_0 \int_{-\frac{\bar{l}_3}{2}}^{\frac{\bar{l}_3}{2}} \int_{-\frac{\bar{l}_2}{2}}^{\frac{\bar{l}_2}{2}} \int_{-\frac{\bar{l}_1}{2}}^{\frac{\bar{l}_1}{2}} s_i s_j ds_1 ds_2 ds_3$$

$$[\bar{\mathbf{J}}] = \frac{\rho_0}{12} \begin{pmatrix} \bar{l}_1^3 \bar{l}_2 \bar{l}_3 & & \\ & \bar{l}_2^3 \bar{l}_3 \bar{l}_1 & \\ & & \bar{l}_3^3 \bar{l}_1 \bar{l}_2 \end{pmatrix}$$

where l_1, l_2, l_3 are the lengths of the edges of the reference shape

If we set $\bar{l}_1 = \bar{l}, \bar{l}_2 = \bar{l}_3 = \alpha \bar{l}$ we get

$$V_{\bar{\mathcal{R}}} = \alpha^2 \bar{l}^3$$

$$[\bar{\mathbb{J}}] = \frac{\rho_0 l^{-2}}{12} \begin{pmatrix} 1 & & \\ & \alpha^2 & \\ & & \alpha^2 \end{pmatrix} \frac{V}{R}$$

let us consider again affine deformations such that

$$[F] = \begin{pmatrix} \lambda & & \\ & \frac{1}{\sqrt{\lambda}} & \\ & & \frac{1}{\sqrt{\lambda}} \end{pmatrix}$$

with λ depending on time, and compute time derivatives

$$[\dot{F}] = \begin{pmatrix} 1 & & \\ & -\frac{1}{2} \lambda^{-\frac{3}{2}} & \\ & & -\frac{1}{2} \lambda^{-\frac{3}{2}} \end{pmatrix} \dot{\lambda}$$

$$[\ddot{F}] = \begin{pmatrix} 1 & & \\ & -\frac{1}{2} \lambda^{-\frac{3}{2}} & \\ & & -\frac{1}{2} \lambda^{-\frac{3}{2}} \end{pmatrix} \ddot{\lambda} + \begin{pmatrix} 0 & & \\ & \frac{3}{4} \lambda^{-\frac{5}{2}} & \\ & & \frac{3}{4} \lambda^{-\frac{5}{2}} \end{pmatrix} \dot{\lambda}^2$$

$$[\ddot{F} F^T] = \begin{pmatrix} \lambda & & \\ & -\frac{1}{2} \lambda^{-2} & \\ & & -\frac{1}{2} \lambda^{-2} \end{pmatrix} \ddot{\lambda} + \begin{pmatrix} 0 & & \\ & \frac{3}{4} \lambda^{-3} & \\ & & \frac{3}{4} \lambda^{-3} \end{pmatrix} \dot{\lambda}^2$$

(71-72)₁₃ Tuesday [2014-05-20] A1.3
9:00 - 11:00

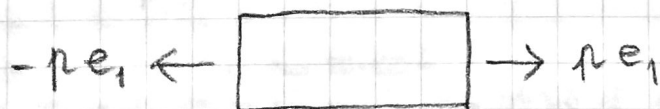
To make expressions simpler let us set

$$\rho_s := \frac{1}{12} \rho_0 \bar{l}^2$$

so we get

$$[\ddot{F} \ddot{J} F^T] = \rho_s \begin{pmatrix} \ddot{\lambda} & 0 & 0 \\ 0 & \alpha^2 \left(-\frac{1}{2} \frac{\ddot{\lambda}}{\lambda^2} + \frac{3}{4} \frac{\dot{\lambda}^2}{\lambda^3} \right) & 0 \\ 0 & 0 & \alpha^2 \left(-\frac{1}{2} \frac{\ddot{\lambda}}{\lambda^2} + \frac{3}{4} \frac{\dot{\lambda}^2}{\lambda^3} \right) \end{pmatrix} V_{\mathbb{R}}$$

Let us apply to the body in the shape of a cylinder with a square cross section a normal traction on the opposite end faces while taking into account the inertial forces



The total force is $f = -m \ddot{p}_0$

The total moment tensor is

$$M = V_{\mathbb{R}} \rho e_1 \otimes e_1 - \ddot{F} \ddot{J} F^T$$

[Notebook page scanned on 2014-06-21]

The balance equations are

$$\dot{f} = 0$$

$$\text{skw } M = 0$$

$$\text{sym } M = T V_R$$

From the first one we get $\dot{p}_0 = 0$

So the barycenter will move at a constant speed.

The moment tensor turns out to be symmetric, as we can see from its matrix. So the second one of the balance equations above is fulfilled.

Since the motion is isochoric

$$V_R = \bar{V}_R$$

Hence, from the last balance equation,

$$T = p e_1 \otimes e_1 - \bar{F} \bar{J} F^T \frac{1}{V_R}$$

whose scalar form is as follows

$$\sigma_1 = \rho - \rho_s \lambda \ddot{\lambda}$$

$$\sigma_2 = 0 - \rho_s \alpha^2 \left(-\frac{1}{2} \frac{\ddot{\lambda}}{\lambda^2} + \frac{3}{4} \frac{\dot{\lambda}^2}{\lambda^3} \right)$$

$$\sigma_3 = 0 - \rho_s \alpha^2 \left(-\frac{1}{2} \frac{\ddot{\lambda}}{\lambda^2} + \frac{3}{4} \frac{\dot{\lambda}^2}{\lambda^3} \right)$$

Let the material be incompressible and viscoelastic

$$\sigma_1 = \hat{\sigma}_1^D - p + \sigma_1^+$$

$$\sigma_2 = \hat{\sigma}_2^D - p + \sigma_2^+$$

$$\sigma_3 = \hat{\sigma}_3^D - p + \sigma_3^+$$

→ (65-66)₁₂

Substituting these expressions into the balance equations we get

$$\sigma_1^D - p + \sigma_1^+ = \rho - \rho_s \lambda \ddot{\lambda}$$

$$\sigma_2^D - p + \sigma_2^+ = -\rho_s \alpha^2 \left(-\frac{1}{2} \frac{\ddot{\lambda}}{\lambda^2} + \frac{3}{4} \frac{\dot{\lambda}^2}{\lambda^3} \right)$$

$$\sigma_3^D - p + \sigma_3^+ = -\rho_s \alpha^2 \left(-\frac{1}{2} \frac{\ddot{\lambda}}{\lambda^2} + \frac{3}{4} \frac{\dot{\lambda}^2}{\lambda^3} \right)$$

By adding terms on the left side and terms on the right side (like taking the trace of the corresponding tensors) we get

$$-3p = \mu - \rho_s \left(\lambda \ddot{\lambda} + \alpha^2 \left(-\frac{\ddot{\lambda}}{\lambda^2} + \frac{3}{2} \frac{\dot{\lambda}^2}{\lambda^3} \right) \right)$$

where

$$\hat{\sigma}_1^D + \hat{\sigma}_2^D + \hat{\sigma}_3^D = 0$$

$$\sigma_1^+ + \sigma_2^+ + \sigma_3^+ = 0$$

because both $\hat{T}(F)$ and T^+ are deviatoric tensors.

We solve the last equation for the inner pressure p and replace its expression in the first of the previous equations

$$\hat{\sigma}_1^D + \sigma_1^+ = \mu - \frac{1}{3}\mu - \rho_s \lambda \ddot{\lambda} + \frac{1}{3}\rho_s \left(\lambda \ddot{\lambda} + \alpha^2 \left(-\frac{\ddot{\lambda}}{\lambda^2} + \frac{3}{2} \frac{\dot{\lambda}^2}{\lambda^3} \right) \right)$$

$$\hat{\sigma}_1^D + 2\mu \frac{\dot{\lambda}}{\lambda} = \frac{2}{3}\mu + \frac{1}{3}\rho_s \left(-2\lambda \ddot{\lambda} + \alpha^2 \left(-\frac{\ddot{\lambda}}{\lambda^2} + \frac{3}{2} \frac{\dot{\lambda}^2}{\lambda^3} \right) \right)$$

$$\hat{\sigma}_1^D = \frac{2}{3}\mu - 2\mu \frac{\dot{\lambda}}{\lambda} - \frac{2}{3}\rho_s \left(\lambda \ddot{\lambda} + \frac{\alpha^2}{2} \left(\frac{\ddot{\lambda}}{\lambda^2} - \frac{3}{2} \frac{\dot{\lambda}^2}{\lambda^3} \right) \right)$$

$$\hat{\sigma}_1^D = \frac{2}{3}\hat{\sigma}_0; \quad \hat{\sigma}_0 = 2c_1 \left(\lambda^2 - \frac{1}{\lambda} \right) \begin{array}{l} \text{neo-Hookean} \\ \text{material} \end{array} \rightarrow (63-64)_{12}$$

$$\hat{\sigma}_0 = \mu - 3\mu \frac{\dot{\lambda}}{\lambda} - \rho_s \left(\lambda \ddot{\lambda} + \frac{\alpha^2}{2} \left(\frac{\ddot{\lambda}}{\lambda^2} - \frac{3}{2} \frac{\dot{\lambda}^2}{\lambda^3} \right) \right)$$

The final expression for the equation of motion is

$$2\kappa_1 \left(\lambda^2 - \frac{1}{\lambda} \right) = \mu - 3\mu \frac{\dot{\lambda}}{\lambda} - \rho_s \left(\lambda \ddot{\lambda} + \frac{\alpha^2}{2} \left(\frac{\dot{\lambda}}{\lambda^2} - \frac{3}{2} \frac{\dot{\lambda}^2}{\lambda^3} \right) \right)$$

We can linearize this equation according to the following procedure

Let us set $\lambda(t) = \lambda_0 + \beta \tilde{\epsilon}(t)$

$$\mu = \mu_0$$

and replace the original expression with a series expansion with respect to β starting at $\beta=0$, up to order 1. Then collect separately terms of order 0 and terms of order 1 with respect to β . We get

$$2\kappa_1 \left(\lambda_0^2 - \frac{1}{\lambda_0} \right) = \mu_0$$

$$4\kappa_1 (1 + \lambda_0^3) \tilde{\epsilon} + 6\lambda_0 \mu \dot{\tilde{\epsilon}} + (\alpha^2 + 2\lambda_0^3) \rho_s \ddot{\tilde{\epsilon}} = 0$$

(73-74)₁₃ Wednesday [2014-05-21] A1.3
9:00-11:00

Summary about the equation of motion and its linearization, describing small oscillations around the solution (λ_0, ρ_0) .

Small oscillations

Let us consider solutions in the general form

$$\tilde{\xi}(t) = \tilde{\xi}_0 e^{kt}$$

The characteristic equation turns out to be

$$4\kappa_1(1+2\lambda_0^3) + (6\lambda_0\mu)k + \rho_s(\alpha^2 + 2\lambda_0^3)k^2 = 0$$

$$\Rightarrow k = \frac{-3\lambda_0\mu \pm \sqrt{9\lambda_0^2\mu^2 - 4\kappa_1\rho_s(1+2\lambda_0^3)(\alpha^2 + 2\lambda_0^3)}}{(\alpha^2 + 2\lambda_0^3)\rho_s}$$

Critical viscosity value μ_0 (depending on λ_0)

$$\mu_0^2 = \frac{2}{3}\kappa_1\rho_s \frac{1}{\lambda_0^2} (1+2\lambda_0^3)(\alpha^2 + 2\lambda_0^3)$$

$\mu < \mu_0$ damped oscillations

$\mu > \mu_0$ overdamping, no oscillations

(75-76)₁₄ Monday [2014-05-26] A1.3
16:00 - 18:00

Linearization around the reference shape

Let us consider a one-parameter family of deformations

$$F(\beta) = R(\beta)U(\beta) \quad \beta \text{ dimensionless}$$

$$R(\beta) = I + \beta \left. \frac{d}{d\beta} R(\beta) \right|_{\beta=0} + o(\beta)$$

$$U(\beta) = I + \beta \left. \frac{d}{d\beta} U(\beta) \right|_{\beta=0} + o(\beta)$$

$$R(\beta)^T R(\beta) = I \quad \Rightarrow \quad \frac{d}{d\beta} (R(\beta)^T R(\beta)) = 0$$

$$\Rightarrow \left. \frac{d}{d\beta} R(\beta)^T \right|_{\beta=0} + \left. \frac{d}{d\beta} R(\beta) \right|_{\beta=0} = 0$$

infinitesimal rotation

$$\Theta := \left. \frac{d}{d\beta} R(\beta) \right|_{\beta=0} \quad \Rightarrow \quad \Theta^T + \Theta = 0$$

infinitesimal stretch

$$E := \left. \frac{d}{d\beta} U(\beta) \right|_{\beta=0} \quad \Rightarrow \quad E^T = E$$

$$F(\beta) = I + \beta \Theta + \beta E + o(\beta)$$

(77-78)_u Tuesday [2014-05-27] A1.3
9:00 - 11:00

Following the formula about time differentiation
already derived \rightarrow (17-18)₃

we get

$$\det F(\beta) = \det F(0) + \beta \left. \frac{d}{d\beta} \det F(\beta) \right|_{\beta=0} + o(\beta)$$

$$\left. \frac{d}{d\beta} \det F(\beta) \right|_{\beta=0} = \det F(0) \operatorname{tr} \left(\left. \left(\frac{d}{d\beta} F(\beta) \right) F(\beta)^{-1} \right) \right|_{\beta=0}$$

$$= \operatorname{tr} \left((\Theta + E)(I - \beta \Theta - \beta E) \right) \Big|_{\beta=0}$$

$$= \operatorname{tr} (\Theta + E) = \operatorname{tr} E$$

$$\det F(\beta) = 1 + \beta \operatorname{tr} E + o(\beta)$$

In view of linearization it is customary to
define the displacement vector field

$$u(\bar{P}_\Delta) := \phi(\bar{P}_\Delta) - \bar{P}_\Delta \quad \forall \bar{P}_\Delta \in \bar{\mathcal{R}}$$

Along any curve \bar{c} we get

$$u(\bar{c}(t)) = \phi(\bar{c}(t)) - \bar{c}(t)$$

$$u(\bar{c}(0)) = \phi(\bar{c}(0)) - \bar{c}(0)$$

$$\frac{u(\bar{c}(h)) - u(\bar{c}(0))}{h} = \frac{\phi(\bar{c}(h)) - \phi(\bar{c}(0))}{h} - \frac{\bar{c}(h) - \bar{c}(0)}{h}$$

Taking the limit as $h \rightarrow 0$ we arrive at

$$\nabla u c' = \nabla \phi c' - c'$$

Hence

$$\nabla u = F - I$$

By using the corresponding series expansions we get

$$\nabla u(\beta) = F(\beta) - I = \beta(\Theta + E) + o(\beta)$$

$$\lim_{\beta \rightarrow 0} \frac{\nabla u(\beta)}{\beta} = \Theta + E$$

Since there is a unique decomposition of any tensor into the sum of a skewsymmetric and a symmetric tensor, by setting

$$\Theta := \text{skw } \nabla u, \quad E := \text{sym } \nabla u$$

we get

$$\nabla u(\beta) = \beta (\text{skw } \nabla u + \text{sym } \nabla u) + o(\beta) = \beta \nabla u + o(\beta)$$

In the linear theory we use quantities like Θ , E , ∇u which are derivatives with respect to β , a dimensionless "scaling factor" for some control parameters like forces.

For the response function of an elastic material we write the series expansion

$$\hat{T}(F(\beta)) = \hat{T}(I) + \beta \left. \frac{d}{d\beta} \hat{T}(F(\beta)) \right|_{\beta=0} + o(\beta)$$

where, because of the objectivity condition

$$\hat{T}(F) = R \hat{T}(U) R^T \quad \rightarrow (37-38)_7$$

we have

$$\left. \frac{d}{d\beta} \hat{T}(F(\beta)) \right|_{\beta=0} = \left. \frac{d}{d\beta} R(\beta) \hat{T}(U(\beta)) R(\beta)^T \right|_{\beta=0}$$

$$= \underbrace{\hat{T}(I)}_0 + \frac{d}{d\beta} \hat{T}(U(\beta)) - \hat{T}(I) \underbrace{0}$$

with

$$\frac{d}{d\beta} \hat{T}(U(\beta)) = \frac{d}{d\beta} \hat{T}(I + \beta E) = \mathbb{C} E$$

where $\mathbb{C} : \text{Sym}(V) \rightarrow \text{Sym}(V)$ is the gradient of of a tensor field over the space of symmetric tensors.

It is referred to as the elasticity tensor.

Notice how

$$\left. \frac{d}{d\beta} \hat{T}(U(\beta)) \right|_{\beta=0} = \lim_{\beta \rightarrow 0} \frac{1}{\beta} (\hat{T}(U(\beta)) - \hat{T}(U(0))) = \mathbb{C} E$$

with

$$E = \left. \frac{d}{d\beta} U(\beta) \right|_{\beta=0} \text{ looks like}$$

(18-20)₄ ←

$$\lim_{h \rightarrow 0} \frac{1}{h} (v(c(h)) - v(c(0))) = \nabla v c'$$

(79-80)₁₄ Wednesday [2014-05-28] 41.3
9:00 - 11:00

Let us recall the definition of the Piola stress

$$S = (\det F) T F^{-T} \quad \rightarrow (39-40)_7$$

Then the first order term in the series expansion of

will be
$$S(\beta) = (\det F(\beta)) T(\beta) F(\beta)^{-T}$$

$$\left. \frac{d}{d\beta} S(\beta) \right|_{\beta=0} = (\operatorname{tr} E) T(0) F(0)^{-T} + (\det F(0)) \left. \frac{d}{d\beta} T(\beta) \right|_{\beta=0} F(0)^{-T} + (\det F(0)) T(0) (\Theta - E)$$

$$T(0) = 0 \quad \Rightarrow \quad \left. \frac{d}{d\beta} S(\beta) \right|_{\beta=0} = \left. \frac{d}{d\beta} T(\beta) \right|_{\beta=0}$$

$$\Rightarrow S(\beta) = T(\beta) + o(\beta)$$

By this property, for a hyperelastic material the relation $\hat{S}(F(\beta)) \cdot \dot{F}(\beta) = \frac{d}{dt} \varphi(F(\beta))$

can be rewritten as

$$o(\beta) + \hat{T}(F(\beta)) \cdot \dot{F}(\beta) = \frac{d}{dt} \varphi(F(\beta))$$

and then, by the results on the previous pages,

$$o(\beta^2) + (\beta \mathbb{C}(E)) \cdot (\beta \Theta + \beta \dot{E}) = \frac{d}{dt} \varphi(U(\beta))$$

where $\varphi(F(\beta)) = \varphi(U(\beta))$ because of objectivity.

By the symmetry of $\mathbb{C}(E)$ we get

$$o(\beta^2) + \beta^2 \mathbb{C}(E) \cdot \dot{E} = \frac{d}{dt} \varphi(U(\beta))$$

For a potential to exist the stress power on the left side should be such that

$$\mathbb{C}(E) \cdot \dot{E} = \mathbb{C}(\dot{E}) \cdot E$$

since $\mathbb{C}: \text{Sym}(V) \rightarrow \text{Sym}(V)$ is a linear function

That means that the elasticity tensor should be symmetric

$$\mathbb{C}E \cdot \dot{E} = E \cdot \mathbb{C}^T \dot{E} = E \cdot \mathbb{C} \dot{E}$$

Hence the strain energy is, up to second order, given by

$$\varphi(E) = \frac{1}{2} \mathbb{C}(E) \cdot E$$

Since $\dim(\text{Sym}(V)) = 6$, the elasticity tensor matrix is a 6×6 matrix. As a symmetric matrix (in an orthonormal tensor basis) it has only 21 independent entries ($\frac{1}{2}(6 \times 6 - 6) + 6$), describing material properties.

It is amazing to find out that by linearizing the general response function for isotropic hyperelastic materials

$$\rightarrow (45-46)_8 \quad \text{we get} \quad \mathbb{C}(E) = \lambda_2 (\text{tr} E) \mathbb{I} + 2 \mu_2 E$$

where, by isotropy, there are only two coefficients left, λ_2 and μ_2 , called the Lamé constants (or moduli).

The symmetry property of the elasticity tensor for a hyperelastic material can be derived through the following detailed computation.

$$\text{Let } \begin{pmatrix} \sigma_{11} \\ \sigma_{12} \\ \sigma_{13} \\ \sigma_{22} \\ \sigma_{23} \\ \sigma_{33} \end{pmatrix} = \begin{pmatrix} c_{11} & c_{12} & c_{13} & c_{14} & c_{15} & c_{16} \\ c_{21} & c_{22} & c_{23} & c_{24} & c_{25} & c_{26} \\ \dots & & & & & \\ \dots & & & & & \\ \dots & & & & & \\ \dots & & & & & \end{pmatrix} \begin{pmatrix} \epsilon_{11} \\ \epsilon_{12} \\ \epsilon_{13} \\ \epsilon_{22} \\ \epsilon_{23} \\ \epsilon_{33} \end{pmatrix}$$

be the component form of the linear elasticity response

$$T = \mathbb{C} E$$

where both T and E are symmetric tensors.

The stress power can be written by using the above components as

$$\begin{aligned} T \cdot \dot{E} &= \mathbb{C}(E) \cdot \dot{E} \\ &= c_{11} \epsilon_{11} \dot{\epsilon}_{11} + c_{12} \epsilon_{12} \dot{\epsilon}_{11} + c_{13} \epsilon_{13} \dot{\epsilon}_{11} + \dots \\ &+ c_{21} \epsilon_{11} \dot{\epsilon}_{12} + c_{22} \epsilon_{12} \dot{\epsilon}_{12} + c_{23} \epsilon_{13} \dot{\epsilon}_{12} + \dots \\ &+ c_{31} \epsilon_{11} \dot{\epsilon}_{13} + c_{32} \epsilon_{12} \dot{\epsilon}_{13} + c_{33} \epsilon_{13} \dot{\epsilon}_{13} + \dots \\ &+ \dots \end{aligned}$$

from which it is clear how $c_{12} = c_{21}$, $c_{13} = c_{31}$, $c_{23} = c_{32}$ etc are the conditions for a primitive to exist.

The linearized response function for an isotropic hyperelastic material can be derived from the general expression $\rightarrow (45-46)_8$ by differentiating $\hat{T}(F(\beta))$ with respect to β .

Since

$$\left. \frac{d}{d\beta} F(\beta) \right|_{\beta=0} = \mathbb{1} + E$$

$$\left. \frac{d}{d\beta} B(\beta) \right|_{\beta=0} = \left. \frac{d}{d\beta} F(\beta) + \frac{d}{d\beta} F(\beta)^T \right|_{\beta=0} = 2E$$

$$\left. \frac{d}{d\beta} B^2(\beta) \right|_{\beta=0} = 4E, \quad \left. \frac{d}{d\beta} B(\beta)^{-1} \right|_{\beta=0} = -2E$$

$$\left. \frac{d}{d\beta} L_3(\beta) \right|_{\beta=0} = \det B(\beta) \operatorname{tr} \left(\left. \frac{d}{d\beta} B(\beta) B(\beta)^{-1} \right|_{\beta=0} \right) = 2 \operatorname{tr} E$$

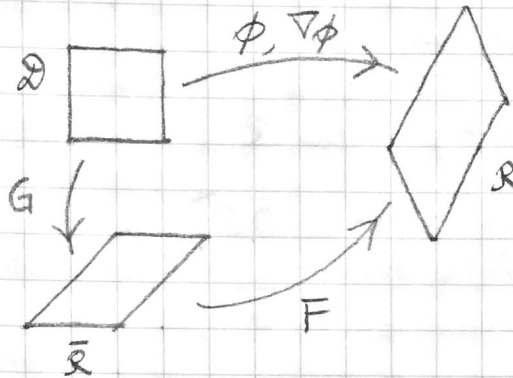
$$\left. \frac{d}{d\beta} L_1(\beta) \right|_{\beta=0} = 2 \operatorname{tr} E, \quad \left. \frac{d}{d\beta} L_2(\beta) \right|_{\beta=0} = 4 \operatorname{tr} E$$

we get an expression as a linear combination of the two tensors $(\operatorname{tr} E)I$ and E which we write

$$\mathbb{C}(E) = \lambda_L (\operatorname{tr} E)I + 2\mu_L E$$

(81-82)₁₅ Tuesday [2014-06-03] Δ1,3
9:00-11:00

Remodeling



$$\nabla\phi = FG \quad \text{Kroner-lee decomposition}$$

F elastic distortion

G remodeling distortion
with $\det G > 0$

\mathcal{D} reference shape

v velocity field on \mathcal{R}

$\bar{\mathcal{R}}$ relaxed shape

V remodeling velocity tensor
field on $\bar{\mathcal{R}}$

\mathcal{R} current shape

φ strain energy density in $\bar{\mathcal{R}}$

Basic assumption: only F affects the strain energy φ .

T Cauchy stress (tensor field on \mathcal{R})

S Piola stress (tensor field on $\bar{\mathcal{R}}$) $S = (\det F) T F^{-T}$

\mathcal{S} Piola stress (tensor field on \mathcal{D}) $\mathcal{S} = (\det \nabla\phi) T \nabla\phi^{-T}$

$$\mathcal{F}^{\text{ext}}(v, V) = \int_{\mathcal{R}} b \cdot r \, dV + \int_{\partial\mathcal{R}} t \cdot r \, dV + \int_{\bar{\mathcal{R}}} \mathcal{Q} \cdot V \, dV$$

\mathcal{Q} remodeling force

$$\mathcal{F}^{\text{int}}(v, V) = - \int_{\mathcal{R}} T \cdot \nabla v \, dV - \int_{\bar{\mathcal{R}}} \mathcal{A} \cdot V \, dV$$

\mathcal{A} remodeling stress

From the force balance principle

$$\mathcal{F}^{\text{ext}}(\nu, V) + \mathcal{F}^{\text{int}}(\nu, V) = 0 \quad \forall \nu, \forall V$$

we get the balance equations

$$\operatorname{div} T + b = 0 \quad \text{in } \mathcal{R}$$

$$Tn = t \quad \text{in } \partial \mathcal{R}$$

$$Q = A \quad \text{in } \bar{\mathcal{R}}$$

In any motion

$$\nabla \nu = \nabla \dot{\phi} \nabla \phi^{-1}$$

$$V = \dot{G} G^{-1}$$

and, by the balance principle above,

$$\int_{\mathcal{R}} b \cdot \nu \, dV + \int_{\partial \mathcal{R}} t \cdot \nu \, dA + \int_{\bar{\mathcal{R}}} Q \cdot \dot{G} G^{-1} \, dV$$

$$= \int_{\mathcal{R}} T \cdot \nabla \dot{\phi} \nabla \phi^{-1} \, dV + \int_{\bar{\mathcal{R}}} A \cdot \dot{G} G^{-1} \, dV$$

Hence the energy balance principle (dissipation inequality)

states that in any motion

$$\int_{\mathcal{R}} T \cdot \nabla \dot{\phi} \nabla \phi^{-1} \, dV + \int_{\bar{\mathcal{R}}} A \cdot \dot{G} G^{-1} \, dV - \frac{d}{dt} \int_{\bar{\mathcal{R}}} \varphi(F) \, dV \geq 0$$

After transforming the integrals into integrals over the reference shape

$$\int_{\mathcal{D}} \mathbf{T} \nabla \phi^{-T} \cdot \nabla \phi \det \nabla \phi \, dV + \int_{\mathcal{D}} \mathbf{A} \cdot \dot{\mathbf{G}} \mathbf{G}^{-1} \det \mathbf{G} \, dV - \frac{d}{dt} \int_{\mathcal{D}} \varphi(\mathbf{F}) \det \mathbf{G} \, dV \geq 0$$

we replace the former statement with the localized form

$$\mathbf{S} \cdot \nabla \phi + \mathbf{A} \cdot \dot{\mathbf{G}} \mathbf{G}^{-1} \det \mathbf{G} - \frac{d}{dt} \varphi(\mathbf{F}) \det \mathbf{G} - \varphi(\mathbf{F}) \det \mathbf{G} \operatorname{tr}(\dot{\mathbf{G}} \mathbf{G}^{-1}) \geq 0$$

where the formula for the time derivative of $\det \mathbf{G}$ has been used $\rightarrow (17-18)_3$

Now let us substitute the relation

$$\hat{\mathbf{S}}(\mathbf{F}) \cdot \dot{\mathbf{F}} = \frac{d}{dt} \varphi(\mathbf{F})$$

together with the expressions for the Piola stress tensors and the Kröner-Lee decomposition, and get in turn

$$\mathbf{S} \cdot (\dot{\mathbf{F}} \mathbf{G} + \mathbf{F} \dot{\mathbf{G}}) \frac{1}{\det \mathbf{G}} + \mathbf{A} \cdot \dot{\mathbf{G}} \mathbf{G}^{-1} - \hat{\mathbf{S}}(\mathbf{F}) \cdot \dot{\mathbf{F}} - \varphi(\mathbf{F}) \mathbf{I} \cdot \dot{\mathbf{G}} \mathbf{G}^{-1} \geq 0$$

$$\frac{\det \nabla \phi}{\det \mathbf{G}} \mathbf{T} \mathbf{F}^{-T} \mathbf{G}^{-T} \cdot (\dot{\mathbf{F}} \mathbf{G} + \mathbf{F} \dot{\mathbf{G}}) - \hat{\mathbf{S}}(\mathbf{F}) \cdot \dot{\mathbf{F}} + (\mathbf{A} - \varphi(\mathbf{F}) \mathbf{I}) \cdot \dot{\mathbf{G}} \mathbf{G}^{-1} \geq 0$$

$$\mathbf{S} \mathbf{G}^{-T} \cdot (\dot{\mathbf{F}} \mathbf{G} + \mathbf{F} \dot{\mathbf{G}}) - \hat{\mathbf{S}}(\mathbf{F}) \cdot \dot{\mathbf{F}} + (\mathbf{A} - \varphi(\mathbf{F}) \mathbf{I}) \cdot \dot{\mathbf{G}} \mathbf{G}^{-1} \geq 0$$

$$\mathbf{S} \cdot \dot{\mathbf{F}} \mathbf{G} \mathbf{G}^{-1} + \mathbf{F}^T \mathbf{S} \cdot \dot{\mathbf{G}} \mathbf{G}^{-1} - \hat{\mathbf{S}}(\mathbf{F}) \cdot \dot{\mathbf{F}} + (\mathbf{A} - \varphi(\mathbf{F}) \mathbf{I}) \cdot \dot{\mathbf{G}} \mathbf{G}^{-1} \geq 0$$

(83-84)₁₅ Wednesday [2014-06-04] A1.3
9:00-11:00

The final expression for the dissipation inequality is

$$(S - \hat{S}(F)) \cdot \dot{F} + (A - \varphi(F)I + F^T S) \cdot \dot{G} G^{-1} \geq 0$$

or, equivalently,

$$\det F (T - \hat{T}(F)) \cdot \dot{F} F^{-1} + \underbrace{(A - (\varphi(F)I - F^T S))}_{\text{Eshelby tensor}} \cdot \dot{G} G^{-1} \geq 0$$

If we set

$$T^+ := T - \hat{T}(F), \quad A^+ := A - (\varphi(F)I - F^T S)$$

the inequality can be written as

$$(\det F) T^+ \cdot \dot{F} F^{-1} + A^+ \cdot \dot{G} G^{-1} \geq 0$$

We can now characterize the dissipation through suitable expressions for T^+ and A^+ consistently with the above inequality in any motion.

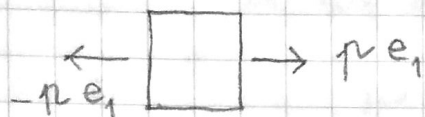
A consistent choice is for example

$$T^+ = 2\mu \operatorname{sym}(\dot{F} F^{-1})$$

$$A^+ = \eta \dot{G} G^{-1} \quad (\eta \geq 0)$$

Note that while T is a symmetric tensor because of the objectivity condition, no such a property has been derived for A .

Let us consider again the body undergoing cylindrical deformations



with

$$[F] = \begin{pmatrix} \lambda & & \\ & \frac{1}{\sqrt{\lambda}} & \\ & & \frac{1}{\sqrt{\lambda}} \end{pmatrix} \quad [G] = \begin{pmatrix} \gamma & & \\ & \frac{1}{\sqrt{\gamma}} & \\ & & \frac{1}{\sqrt{\gamma}} \end{pmatrix}$$

Hence

$$[\nabla\phi] = \begin{pmatrix} \lambda\gamma & & \\ & \frac{1}{\sqrt{\lambda\gamma}} & \\ & & \frac{1}{\sqrt{\lambda\gamma}} \end{pmatrix}$$

Balance equations

$$f = 0$$

$$\text{skw } M = 0$$

$$\frac{M}{V_R} = T$$

$$Q = A$$

where

$$M = r e_1 \otimes e_1 V_R$$

Let us set the remodeling force

$$\mathcal{Q} = 0$$

From the balance equations we get

$$\mathbf{T} = p \mathbf{e}_1 \otimes \mathbf{e}_1$$

$$\mathbf{A} = 0$$

Material characterization

$$\mathbf{T} = \hat{\mathbf{T}}(\mathbf{F}) - p \mathbf{I} + \mathbf{T}^+$$

$$\mathbf{A} = \varphi(\mathbf{F}) \mathbf{I} - \mathbf{F}^T \mathbf{S} - a \mathbf{I} + \mathbf{A}^+$$

where the inner pressure p is the reactive stress

defined by the property that its power is zero, because $\dot{\mathbf{F}}\mathbf{F}^{-1}$ is deviatoric ($\text{tr} \dot{\mathbf{F}}\mathbf{F}^{-1} = 0$) here.

Since $\dot{\mathbf{G}}\mathbf{G}^{-1}$ is deviatoric as well, because of the assumption of isochoric remodeling, we have to add the remodeling pressure a , whose power is $a \mathbf{I} \cdot \dot{\mathbf{G}}\mathbf{G}^{-1} = 0$, because $\text{tr} \dot{\mathbf{G}}\mathbf{G}^{-1} = 0$ here.

Let us set now $\mathbf{T}^+ = 0 \quad (\Leftrightarrow \mu = 0)$

$$\mathbf{A}^+ = \eta \dot{\mathbf{G}}\mathbf{G}^{-1} \quad (\eta > 0)$$

and choose a neo-Hookean strain energy

$$\varphi(\mathbf{F}) = c_1 (I_1 - 3)$$

(85-86)₁₆ Monday [2014-06-09] A1.3
16:00-18:00

$$T = \hat{T}(F) - pI$$

$$A = \varphi(F)I - F^T T F^{-T} \det F - aI + \eta \dot{G} G^{-1}$$

$$\det F = 1$$

$$[F^T T F^{-T}] = \begin{pmatrix} \lambda & & \\ & \frac{1}{\sqrt{\lambda}} & \\ & & \frac{1}{\sqrt{\lambda}} \end{pmatrix} \begin{pmatrix} \sigma_1 & & \\ & \sigma_2 & \\ & & \sigma_3 \end{pmatrix} \begin{pmatrix} \frac{1}{\lambda} & & \\ & \sqrt{\lambda} & \\ & & \sqrt{\lambda} \end{pmatrix} = [T]$$

$$[T] = \begin{pmatrix} \sigma_1 & & \\ & \sigma_2 & \\ & & \sigma_3 \end{pmatrix} \quad [A] = \begin{pmatrix} a_1 & & \\ & a_2 & \\ & & a_3 \end{pmatrix}$$

$$[\dot{G} G^{-1}] = \begin{pmatrix} 1 & & \\ & -\frac{1}{2} & \\ & & -\frac{1}{2} \end{pmatrix} \begin{pmatrix} \dot{\gamma} \\ \dot{\gamma} \\ \dot{\gamma} \end{pmatrix} \quad [\dot{F} F^{-1}] = \begin{pmatrix} 1 & & \\ & -\frac{1}{2} & \\ & & -\frac{1}{2} \end{pmatrix} \begin{pmatrix} \dot{\lambda} \\ \dot{\lambda} \\ \dot{\lambda} \end{pmatrix}$$

$$[C] = [F^T F] = \begin{pmatrix} \lambda^2 & & \\ & \frac{1}{\lambda} & \\ & & \frac{1}{\lambda} \end{pmatrix}$$

$$b_1 = b_2 C = \lambda^2 + \frac{2}{\lambda}$$

[Notebook page scanned on 2014-06-16]

stress characterization

$$\sigma_1 = \hat{\sigma}_1^D - p$$

$$\sigma_2 = \hat{\sigma}_2^D - p$$

$$\sigma_3 = \hat{\sigma}_3^D - p$$

$$T = \hat{T}(F) - pI$$

balance

$$\sigma_1 = \mu$$

$$\sigma_2 = 0$$

$$\sigma_3 = 0$$

$$T = \frac{M}{V_R}$$

$$\begin{aligned} \hat{\sigma}_1^D - p &= \mu \\ \hat{\sigma}_2^D - p &= 0 \\ \hat{\sigma}_3^D - p &= 0 \end{aligned}$$

trace

$$0 - 3p = \mu$$

$$\Rightarrow p = -\frac{1}{3}\mu$$

$$\hat{\sigma}_1^D = \mu + p = \frac{2}{3}\mu$$

$$\hat{\sigma}_2^D = p = -\frac{1}{3}\mu$$

$$\hat{\sigma}_3^D = p = -\frac{1}{3}\mu$$

response function

$$\hat{T}(F) \cdot \dot{F}F^{-1} = \left(\hat{\sigma}_1 - \frac{1}{2}(\hat{\sigma}_2 + \hat{\sigma}_3) \right) \frac{\dot{\lambda}}{\lambda} \quad \hat{\sigma}_0 := \hat{\sigma}_1 - \frac{1}{2}(\hat{\sigma}_2 + \hat{\sigma}_3)$$

for a neo-Hookean material

$$\frac{d}{dt} \varphi(F) = 2c_1 \left(\lambda^2 - \frac{1}{\lambda} \right) \frac{\dot{\lambda}}{\lambda} \Rightarrow \hat{\sigma}_0 = 2c_1 \left(\lambda^2 - \frac{1}{\lambda} \right)$$

Because of incompressibility only the deviatoric part of the response $\hat{T}(F)$ is delivered by the strain energy.

Hence
$$\hat{\sigma}_1^D = \hat{\sigma}_1 - \frac{1}{3}(\hat{\sigma}_1 + \hat{\sigma}_2 + \hat{\sigma}_3) = \frac{2}{3}\left(\hat{\sigma}_1 - \frac{1}{2}(\hat{\sigma}_2 + \hat{\sigma}_3)\right)$$

$$\Rightarrow \hat{\sigma}_1^D = \frac{2}{3}\hat{\sigma}_0$$

$$\hat{\sigma}_1^D = \frac{2}{3}\mu \quad (\text{balance \& stress characterization})$$

$$\frac{2}{3}\hat{\sigma}_0 = \frac{2}{3}\mu$$

$$2c_1\left(\lambda^2 - \frac{1}{\lambda}\right) = \mu$$

balance with
neo-Hookean
response

The deformation gradient depends on both λ and γ .

remodeling stress

balance

$$a_1 = \varphi(F) - \sigma_1 - a + \eta \frac{\dot{\gamma}}{\gamma}$$

$$a_1 = 0$$

$$a_2 = \varphi(F) - \sigma_2 - a - \frac{1}{2}\eta \frac{\dot{\gamma}}{\gamma}$$

$$a_2 = 0$$

$$a_3 = \varphi(F) - \sigma_3 - a - \frac{1}{2}\eta \frac{\dot{\gamma}}{\gamma}$$

$$a_3 = 0$$

$$\rightarrow \varphi(F) - \hat{\sigma}_1^D + p - a + \eta \frac{\dot{\gamma}}{\gamma} = 0 \quad \leftarrow$$

$$\varphi(F) - \hat{\sigma}_2^D + p - a - \frac{1}{2}\eta \frac{\dot{\gamma}}{\gamma} = 0$$

$$\varphi(F) - \hat{\sigma}_3^D + p - a - \frac{1}{2}\eta \frac{\dot{\gamma}}{\gamma} = 0$$

$$3\varphi(F) + 3p - 3a = 0$$

$$\Rightarrow a = \varphi(F) + p$$

$$\cancel{\varphi(F)} - \frac{2}{3} \cancel{\mu} + p - \cancel{\varphi(F)} - p + \eta \frac{\dot{\gamma}}{\gamma} = 0$$

$$\cancel{\varphi(F)} - \cancel{\varphi(F)} - p - \frac{1}{2} \eta \frac{\dot{\gamma}}{\gamma} = 0$$

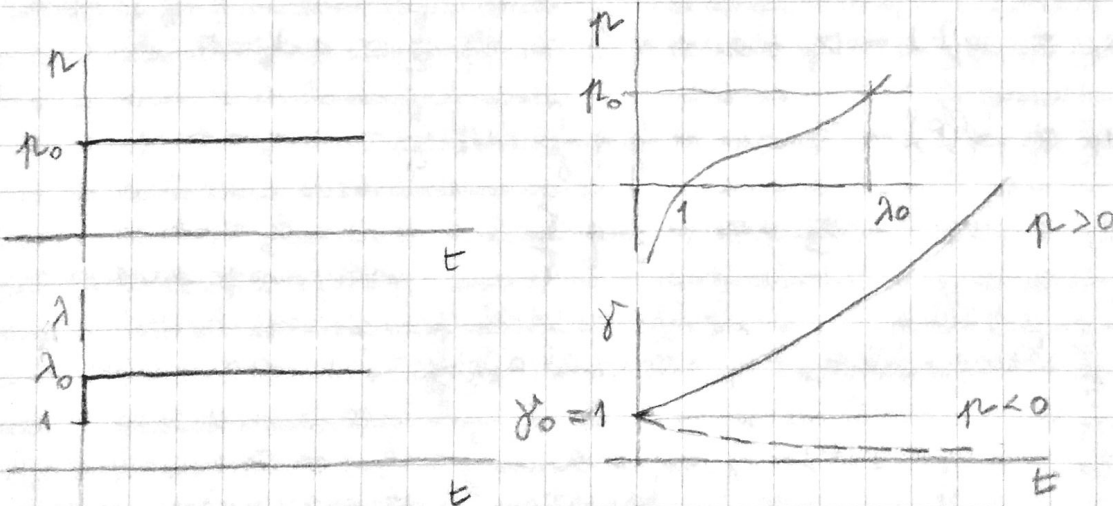
$$\cancel{\varphi(F)} - \cancel{\varphi(F)} - p - \frac{1}{2} \eta \frac{\dot{\gamma}}{\gamma} = 0$$

From the first equation we get

$$\textcircled{1} \quad \eta \frac{\dot{\gamma}}{\gamma} = \frac{2}{3} \mu$$

$$\gamma(t) = \gamma_0 e^{kt}$$

$$k = \frac{1}{\eta} \frac{2}{3} \mu$$



$$\lambda(t) \gamma(t) = \lambda_0 \gamma(t) \begin{cases} \rightarrow \infty & \mu > 0 \\ \rightarrow 0 & \mu < 0 \end{cases}$$

with $\mu(t) = \mu_0$ and $\lambda(t) = \lambda_0$

We can address a different problem within the same setting.

Let the body be constrained to keep its length unchanged after stretching it at time $t=0$

by $\lambda(0) = \lambda_0$, with $y(0) = 1$, so that

$$\lambda(t)y(t) = \lambda_0 \quad t \geq 0$$

Let p_0 be the force needed to get such a stretch.

If λ changes in time the force will change according to

$$2c_1 \left(\lambda(t)^2 - \frac{1}{\lambda(t)} \right) = p(t) \quad (\text{neo-Hookean material})$$

Therefore

$$\eta \frac{\dot{y}(t)}{y(t)} = \frac{2}{3} p(t)$$

becomes

$$\eta \frac{\dot{y}(t)}{y(t)} = \frac{4}{3} c_1 \left(\lambda(t)^2 - \frac{1}{\lambda(t)} \right)$$

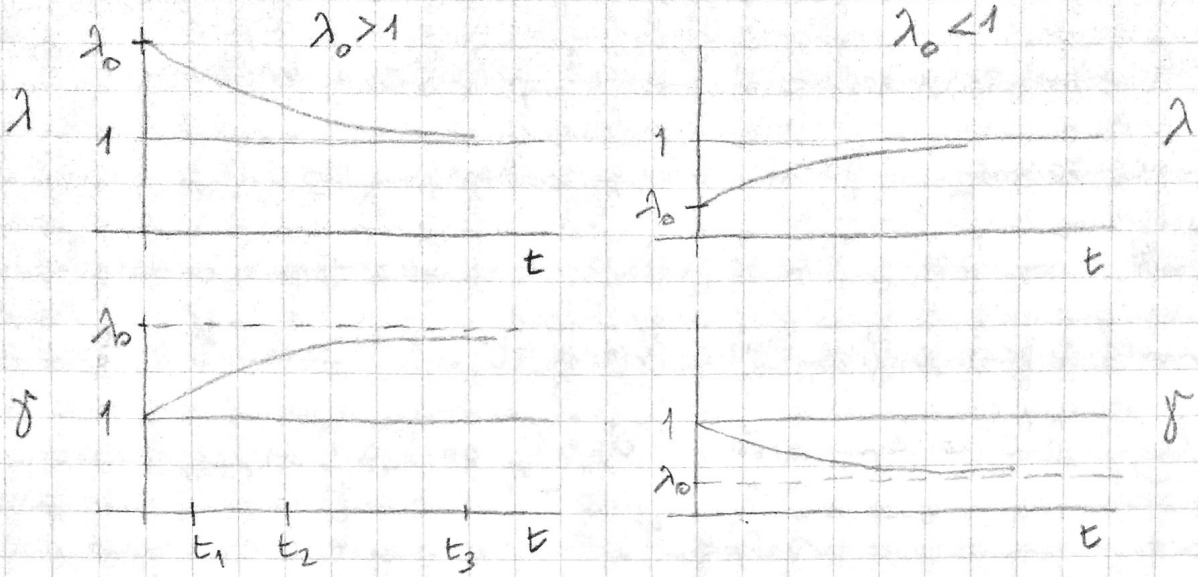
From the constraint $\lambda(t)y(t) = \lambda_0$ we get

$$\dot{\lambda}(t)y(t) + \lambda(t)\dot{y}(t) = 0$$

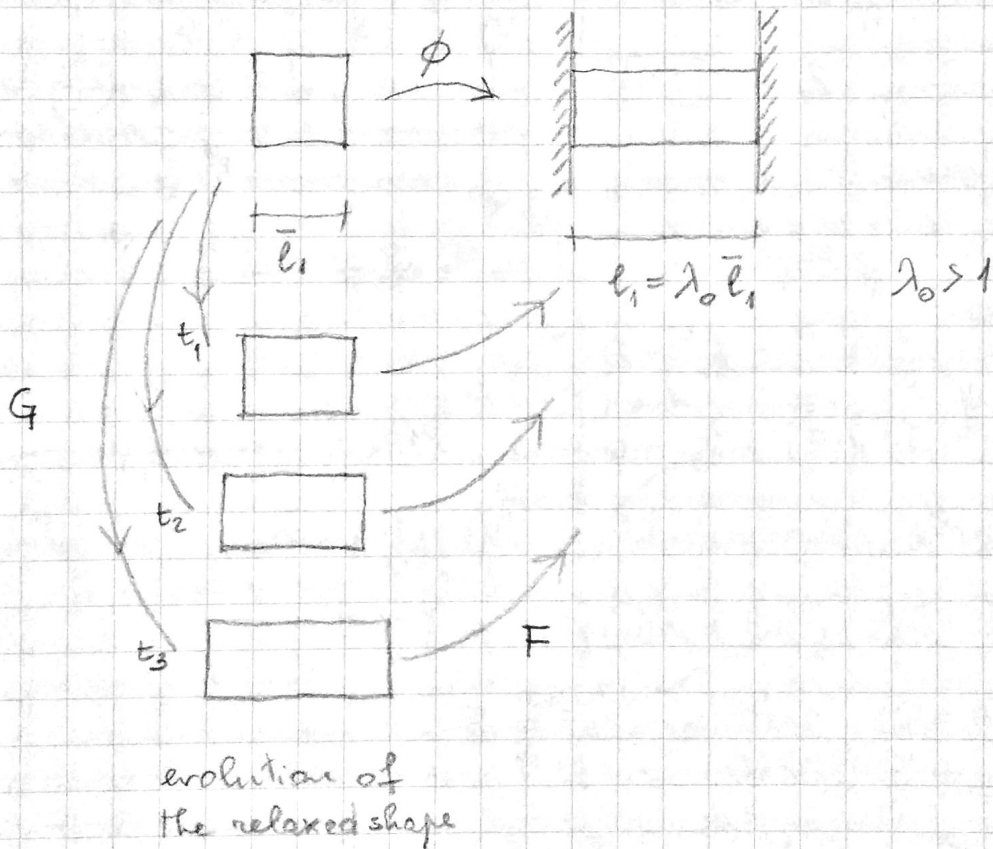
$$\frac{\dot{\lambda}(t)}{\lambda(t)} = - \frac{\dot{y}(t)}{y(t)}$$

Hence the evolution of the elastic stretch is described by

$$\textcircled{2} \quad \eta \frac{\dot{\lambda}}{\lambda} = - \frac{4}{3} c_1 \left(\lambda^2 - \frac{1}{\lambda} \right) \quad \text{with } \lambda(0) = \lambda_0$$



Note that the only stationary solution is $\lambda = 1$



(87-88)₁₆ Wednesday [2014-06-11] A1.3
9:00 - 11:00

Let us now consider a case where the remodeling force is different from zero

$$\mathcal{Q} = q \mathbf{e}_1 \otimes \mathbf{e}_1$$

From the balance equation

$$\mathbf{A} = \mathcal{Q}$$

we get

$$a_1 = q$$

$$a_2 = 0$$

$$a_3 = 0$$

and, from the remodeling stress characterization,

$$\varphi(F) - \sigma_1^D + p - a + \eta \frac{\dot{\sigma}}{\gamma} = q$$

$$\varphi(F) - \sigma_2^D + p - a - \frac{1}{2} \eta \frac{\dot{\sigma}}{\gamma} = 0$$

$$\varphi(F) - \sigma_3^D + p - a - \frac{1}{2} \eta \frac{\dot{\sigma}}{\gamma} = 0$$

$$3\varphi(F) + 3p - 3a = q$$

$$\Rightarrow a = \varphi(F) + p - \frac{1}{3}q$$

Substituting this expression into the equations above

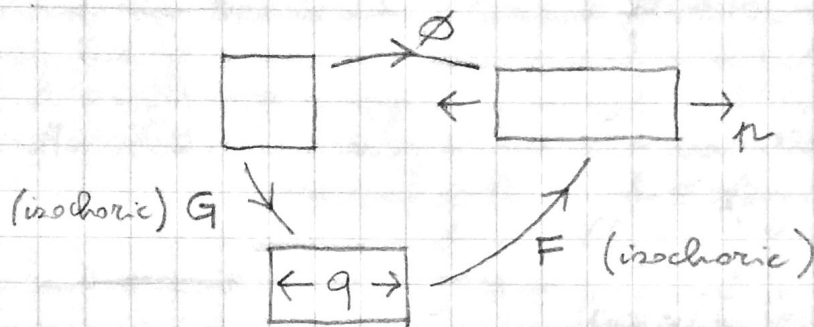
we get

$$\cancel{\varphi(F)} - \frac{2}{3}p + p - \cancel{\varphi(F)} - p + \frac{1}{3}q + \eta \frac{\dot{\sigma}}{\gamma} = q$$

which simplifies to

$$\textcircled{3} \quad \eta \frac{\dot{\gamma}}{\gamma} = \frac{2}{3} (\mu + q)$$

Note that the remodeling force q can either increase or decrease the effect of μ



Let us now consider the body constrained to keep its length unchanged while under the action of q so that $\lambda(t) \gamma(t) = \lambda_0$, with $\gamma(0) = 1$.

Since $\frac{\dot{\gamma}}{\gamma} = -\frac{\dot{\lambda}}{\lambda}$, by using the expression for μ derived from a neo-Hookean strain energy we get

$$\textcircled{4} \quad \eta \frac{\dot{\lambda}}{\lambda} = -\frac{2}{3} \left(2c_1 \left(\lambda^2 - \frac{1}{\lambda} \right) + q \right)$$

There is a unique stationary solution for each value of q , as shown by the graph $(\lambda, q) \equiv (\lambda, -\mu)$

In order to describe a volumetric growth let us consider a case where G is a spherical tensor

$$[G] = \begin{pmatrix} \delta & & \\ & \delta & \\ & & \delta \end{pmatrix}$$

while F is still isochoric

$$[F] = \begin{pmatrix} \lambda & & \\ & \frac{1}{\sqrt{\lambda}} & \\ & & \frac{1}{\sqrt{\lambda}} \end{pmatrix} \quad [\nabla\phi] = \begin{pmatrix} \lambda\delta & & \\ & \frac{\delta}{\sqrt{\lambda}} & \\ & & \frac{\delta}{\sqrt{\lambda}} \end{pmatrix}$$

The remodeling velocity will be a spherical tensor

$$\dot{G}G^{-1} = \frac{\dot{\delta}}{\delta} \mathbf{I}$$

That is why we assume the remodeling force be described by a spherical tensor as well

$$\mathbf{Q} = q \mathbf{I}$$

The scalar form of the remodeling stress characterization is

$$a_1 = \varphi(F) - \sigma_1 + a_0 + \eta \frac{\dot{\delta}}{\delta}$$

$$a_2 = \varphi(F) - \sigma_2 - \frac{1}{2} a_0 + \eta \frac{\dot{\delta}}{\delta}$$

$$a_3 = \varphi(F) - \sigma_3 - \frac{1}{2} a_0 + \eta \frac{\dot{\delta}}{\delta}$$

where a_0 has been used to describe the deviatoric part of the remodeling stress whose power is zero since the remodeling velocity is now a spherical tensor.

Substituting the balance equations

$$\sigma_1 = \mu$$

$$a_1 = q$$

$$\sigma_2 = 0$$

$$a_2 = q$$

$$\sigma_3 = 0$$

$$a_3 = q$$

we get

$$\varphi(F) - \mu + a_0 + \eta \frac{\dot{\gamma}}{\gamma} = q$$

$$\varphi(F) - \frac{1}{2} a_0 + \eta \frac{\dot{\gamma}}{\gamma} = q$$

$$\varphi(F) - \frac{1}{2} a_0 + \eta \frac{\dot{\gamma}}{\gamma} = q$$

$$3\varphi(F) - \mu + 3\eta \frac{\dot{\gamma}}{\gamma} = 3q$$

$$\Rightarrow \textcircled{5} \quad \eta \frac{\dot{\gamma}}{\gamma} = - \left(\varphi(F) - \frac{1}{3} \mu \right) + q$$

and, from any of the equations above,

$$a_0 = \frac{2}{3} \mu$$

For a neo-Hookean material

$$\varphi(F) = c_1 (I_1 - 3) = c_1 \left(\lambda^2 + \frac{2}{\lambda} - 3 \right)$$

$$p = 2c_1 \left(\lambda^2 - \frac{1}{\lambda} \right)$$

$$\varphi(F) - \frac{1}{3}p = \frac{1}{3}c_1 \left(\lambda^2 + \frac{8}{\lambda} - 9 \right)$$

Let us consider again the case where the body is constrained between two walls in such a way that

$$\lambda(t) \gamma(t) = \lambda_0 \quad t \geq 0$$

with $\lambda(0) = \lambda_0$ and $\gamma(0) = 1$.

From the condition above we get again

$$\frac{\dot{\lambda}}{\lambda} = -\frac{\dot{\gamma}}{\gamma}$$

leading to the evolution equation for the elastic stretch

$$\textcircled{6} \quad \eta \frac{\dot{\lambda}}{\lambda} = \frac{1}{3}c_1 \left(\lambda^2 + \frac{8}{\lambda} - 9 \right) - q$$